

Effect of Manufacturing Inaccuracies to the Wake past Asymmetric Airfoil by PIV

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Abstract: The effect of manufacturing geometry deviations to the flow past NACA 64(3)-618 asymmetric airfoil is studied. This airfoil is 3D printed according to the coordinates from public database. An optical high-precision 3D scanner GOM Atos measures the difference from the idealized model. Based on this difference, another model is prepared with physical output closer to the ideal model. The velocity in the near wake (0 – 0.4 chord) is measured by using the Particle Image Velocimetry (PIV) technique. This work compares the wakes past three airfoil realizations, which differ in their similarity to the original design (none of the realizations is identical to the original design). The chord-based Reynolds number ranges from $1.6 \cdot 10^4$ to $1.6 \cdot 10^5$. The ensemble average velocity is used for determination of the wake width and for roughly estimation of the drag coefficient. The lift coefficient is measured directly by using force balance. We discuss the origin of turbulent kinetic energy in terms of anisotropy (at least in 2D) and the length-scales of fluctuations across the wake. The spatial power spectral density is shown. Autocorrelation function of cross-stream velocity detects the regime of von Kármán vortex street at lower velocities.

Keywords: Particle Image Velocimetry ; Wake ; 3D scanning ; NACA 64-618 ; Turbulent kinetic energy ; Spectrum

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1. Introduction

The fluid flow is a highly non-linear problem without reasonable solution up to nowadays. One of the general effects of non-linearity is the unpredictable response to even a small perturbations or changes of the boundary conditions. The scale of possible responses to a small geometry perturbation ranges from almost zero effect, through a linear response up to a complex change of flow state. The famous *butterfly effect* represents this behavior on the example of a small butterfly which *can* switch the evolution path of the turbulent system [1]. As there is a large number of such micro-events, the evolution path of the entire system is unpredictable. But, the statistical properties can be predicted quite reasonably. This feature is used in modern *computational fluid dynamics*, which does not solve the non-linear Navier-Stokes equations on a fine mesh with resolution of Kolmogorov length-scale, but it solves only much larger cells with direct link to the geometry of the boundary conditions under assumption, that the behavior at smaller scales follows some of the *turbulence models*.

The relatively low-cost computational fluid dynamics attacks the position of thorough experiments on the place of industrial design and optimization. But the computational methods need validation and verification [2], which is mainly based on the comparison with an appropriate experiment, which does not need to fit in all parameters, but, at least, the main geometrical and fluid properties might be met. And here comes

36 the problem with geometry – the object used in the experimental study is different from
37 the desired design used in the computational approach. As the quality needs to increase,
38 even smaller deviations between numerical and experimental results can be accepted.
39 As the experiment has an aura of the truth, the mathematicians sweat blood developing
40 even better and more accurate models in order to fit the experimental data measured
41 with boundary conditions known only approximately.

42 In this study, we use a high-quality contact-less 3D optical scanner to measure the
43 real geometry of boundary conditions (of course, with some uncertainty connected with
44 each measurement technique) and compare the flow field past an airfoil produced at
45 three levels of accuracy – first variant represents the naive approach, we send to the 3D
46 printer just the coordinates obtained from public database [3]. If we have not got the 3D
47 optical scanner, our study ended here declaring "we measured flow in the near wake
48 past airfoil...". However, the 3D scan discovered discrepancies from the ideal geometry,
49 therefore we try to prevent them by manipulating the geometry in a such way, that the
50 outcome is closer to the ideal case. This step is repeated creating two more variants
51 closer and closer to the ideal geometry. Anyway, the best product is still different from
52 the ideal geometry used in the processes of numerical optimization.

53 There is only very little other studies about the manufacturing inaccuracies and they
54 are numerical only. Our study is experimental and it compares really created models. A
55 disadvantage is, that there does not exist the ideal geometry in reality, hence we can
56 compare only different levels of manufacturing attention.

57 1.1. Some recent literature concerning this problem

58 Moreno-Oliva and coworkers [4] used the Laser Triangulation Technique to measure
59 the real profile of a NACA0012 and FX 61-137 airfoil, one realization of the NACA0012
60 has been commercially made of metal, the second has been 3D printed by the authors
61 from PLA, the profile FX 61-137 was measured at real wind turbine blade at several
62 heights. They used the measured profiles as an input into the Xfoil software [5] together
63 with the theoretical profiles. They report a significant decrease of performance of the
64 real wind turbine profiles when compared with the ideal geometry.

65 Ravikovich et al. [6] performed an optimization of fan blades with real geometry
66 deviations from airfoil geometry. Their algorithm simultaneously solves mechanical
67 integrity problems and aerodynamic stability of the flow. The geometry data measured
68 at cold conditions are transformed via numerical simulation into hot conditions, which
69 allows to estimate the influence of measured geometry to final fan parameters.

70 The effect of even small geometry modifications in turbine stator blades is studied
71 by Klimko and Okresa [7]. The modifications in stator wheel affects the reaction and
72 efficiency of the entire stage. Data like the just mentioned ones are used in numerical
73 optimization [8] of developed new stage.

74 Winstroth and Seume [9] studied numerically the effect of several artificial geome-
75 try deviations on four wind turbine airfoils. The geometry deviations cover mold tilt
76 towards leading or trailing edge, step changes at certain position, sine wave placed at
77 several positions on both pressure and suction side or at leading edge, and thickening of
78 the profile – in total 40 modifications of four different airfoil profiles used in the studied
79 wind turbine. They found that the worst deviation (in terms of energy production) is the
80 tilt towards leading edge; they found also some cases with positive impact.

81 The structure of our article is as follows: first we describe the methods including
82 3D printing, 3D optical scanning and Particle Image Velocimetry used to measure 2D
83 velocities within a square area past the trailing edge. This paper focuses to data at zero
84 angle of attack, some data at angle of attack $\alpha = 10^\circ$ are in the appendix. The ensemble
85 average stream-wise velocity, the path of wake centerline and the wake width based on
86 average stream-wise velocity are shown. The lift coefficient is impossible to determine on
87 the basis of PIV data, therefore a simple balance is introduced and the lift at zero angle
88 of attack and at several Reynolds number is measured. The turbulent kinetic energy

89 is explored discussing, if it is produced by fluctuations in stream-wise or cross-stream
90 velocity component. The length-scale of fluctuations is explored by unique method
91 developed at the University of West Bohemia in Pilsen. The autocorrelation function
92 reveals the periodicity at low velocities and the decay of integral length-scale at larger
93 ones.

94 2. Materials and methods

95 2.1. 3D printer

96 We used a commercially available 3D printer Prusa Mk 2.5. It uses the FDM (Fused
97 Deposition Modeling) printing technology [10]. The used material is the thermoplastic
98 polymerized lactic acid PLA [11]. The nozzle temperature was 230 °C, the height of a
99 single layer 0.15 mm. The 3D printed airfoils are used mainly for testing and develop-
100 ment purposes. At the current stage, 3D printing technology is not suitable for industrial
101 manufacturing. However, it is important to know the impact of the typical 3D printing
102 shortcomings.

103 2.2. 3D scanner

104 The optical 3D scanning method FPS (Fringe Projection Scanning [12]) is based
105 on projecting light periodic stripes onto the surface [12,13] and the variations of their
106 phase is used to estimate the variations of height across the measured surface. The
107 commercial 3D optical scanner GOM Atos Core [14] is used in our study. Mendřický [15]
108 measured accuracy of a such scanner in dependence on the calibration and the optical
109 matteness of measuring surface of standard spheres. He found that the accuracy is in
110 order of micrometers, matte coating adds around 10^{-5} m, while just an old calibration
111 can change measured results systematically by around $2 \cdot 10^{-5}$ m. Li et al. [13] found
112 that the ambient light has only small effect on the results, while Vagovský et al. [16]
113 states that even the size of scanned object influences the accuracy due to the focus blur.
114 Measuring volume of our scanner is 300 cm³, thus the airfoil of height 12.5 cm is small
115 for such volume, but still, our object size is much closer to the measured volume than
116 in the case explored by Vagovský [16]. We did not used the matte coating because we
117 used white PLA material; on the other hand, we had to paint it to black before the PIV
118 measurement, which uses intense laser light, and we did not measured the thickness
119 added by this painting.

120 2.3. Model

121 The coordinates of NACA 64-618 airfoil were got from a public database Airfoil-
122 tools.com [3]. This airfoil is frequently used at tips of the wind turbines [17] and have
123 a reasonable lift even at high angles of attack (which is valid only for larger Reynolds
124 numbers, as will be shown later). The trailing edge of this airfoil is sharp and it points
125 towards the high pressure side, see the black line in Figure 1. First, this airfoil was 3D
126 printed just according the point coordinates obtained from the public database [3]; this
127 first naive realization is denoted A. Then a high-precision 3D optical scanner GOM Atos
128 [18] is used to measure the deviation from the ideal geometry, see Figure 1. There exists
129 a huge systematic printing error – the missing trailing edge, see Figure 1c. This is caused
130 by the finite thickness of the extrusion line, which does not fit into the too thin walls
131 and the program for generating the g-code does not let the extruder moving into that
132 locations. The trailing edge is shorter by almost 3 mm, which is 3.7% of the chord length!

133 A next step was to create the trailing edge thicker in order to fill it with at least some
134 material. This was done by Minkowski sum [19] of the airfoil profile with an circle of
135 radius 0.2 mm. This way leads to too large thickening of the entire model, but the trailing
136 edge is still shorter than it should be. This blind way is not reported here. Next trial,
137 denoted B, is obtained by artificial manipulation with the profile points at the trailing
138 edge combined with the Minkowski sum with circle of radius 0.1 mm. Now, the trailing

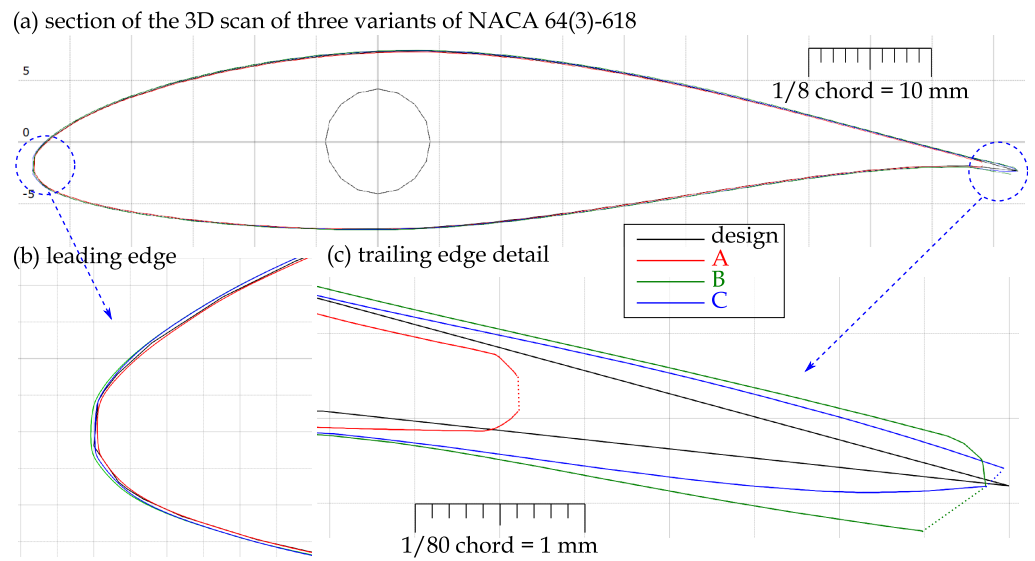


Figure 1. Comparison of 3D scans of three manufactured realizations of the NACA 64-618 airfoil, which is depicted by solid black line. The production details are in text. Panels (b) and (c) zoom the details of leading and trailing edge respectively.

Table 1: Dimensions of the studied airfoil realizations compared with the ideal case denoted CAD in first column. The blockage ratio of the wind tunnel is the ratio of blocked projection area to the total cross-section area (125×125 mm), it is calculated from the *caliper* approach, i.e. the distance of plane touching suction side to second plane touching the pressure side, when their normals are locked to the y -axis.

	Unit	CAD	A	B	C
Chord length c	[mm]	80.00	77.01	79.83	80.08
Profile thickness – maximum inscribed circle	[mm]	14.36	14.34	14.55	14.47
Profile thickness – caliper along y -axis	[mm]	14.44	14.41	14.63	14.55
Blockage ratio	[%]	11.55	11.53	11.70	11.64

139 edge is shorter by around 0.2 mm only. On the other hand, the trailing edge displays a
 140 large radius of around 0.3 mm.

141 The best result is obtained by using Minkowski sum with circle of 0.1 mm radius,
 142 adding new points into the trailing edge area expanding it over the ideal edge. After
 143 printing, the model was polished manually in order to sharp the trailing edge. As a side
 144 effect, the surface of this last realization, denoted C, is smoother than the previous ones.
 145 The problem with manual polishing is that (i) it is a non-repeatable procedure, and (ii) it
 146 is too expensive for possible industrial use.

147 2.4. Wind tunnel

148 University of West Bohemia in Pilsen has got multiple wind tunnels [20,21]. But,
 149 due to the laser safety requirements, we are limited with the use of PIV method to
 150 the "small" wind tunnel [22] with transparent test-section of length 400 mm, and cross-
 151 sectional size 125×125 mm. It is an open low-speed wind tunnel driven by the radial
 152 fan at the tunnel inlet. The smallest stable velocity used in this study is $U_{ref}^{min} = 3.1$ m/s;
 153 the largest used reference velocity is 30.6 m/s. The working fluid is atmospheric air of
 154 temperature around 20°C , the actual humidity was not measured.

155 The Reynolds number used in this study is based on the above mentioned reference
 156 velocity U_{ref} , which fits the incoming velocity in the case of empty test section of the

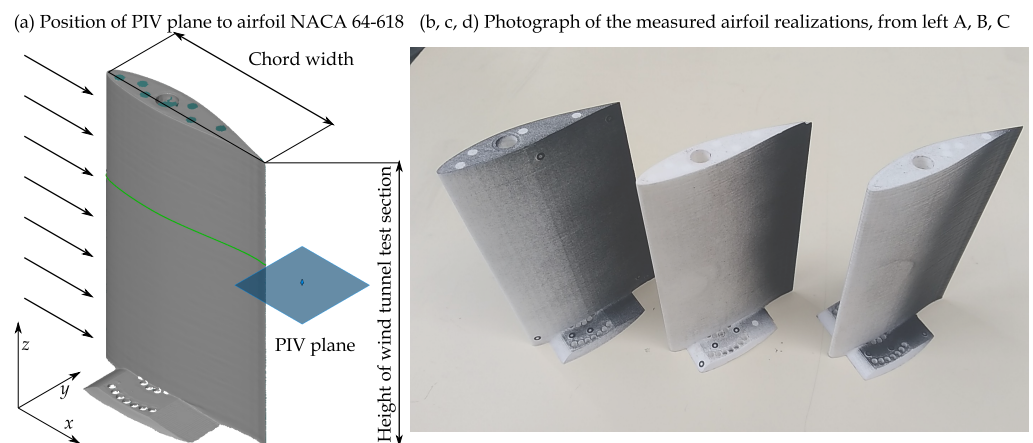


Figure 2. (a) Position of the area studied by using Particle Image Velocimetry depicted as a blue square just behind the trailing edge of the airfoil. The PIV area is a square of side $32 \text{ mm} = 0.4c$. The estimated thickness of the illuminated plane is $\sim 1 \text{ mm}$. The laser light comes from counter stream direction. (b - d) Photograph of the 3D printed realizations of the airfoil. Their trailing parts are blacked in order to suppress the laser reflections.

157 wind tunnel. As the length-scale for Reynolds number, the planned chord length of
 158 the airfoil is used. It is a constant $c = 80 \text{ mm}$. Note, the real chord slightly differs
 159 among realizations A, B and C (see Table 1 for more details). The range of Reynolds
 160 numbers used in this study spans one order of magnitude from $1.63 \cdot 10^4$ (corresponding
 161 to $U_{\text{ref}} = 3.1 \text{ m/s}$) up to $1.63 \cdot 10^5$, where $U_{\text{ref}} = 30.6 \text{ m/s}$.

162 2.5. Particle Image Velocimetry

163 The method Particle Image Velocimetry [23] visualizes the wake structure in a small
 164 area just behind the airfoil, see Figure 2(a). This method is based on optical observation
 165 of movement of small particles carried by the studied fluid. In our case, the particles
 166 are droplets of Safex of diameter in the order of micrometers. When the droplets are
 167 small enough, they follow the fluid motion because the inertial forces scale with particle
 168 volume, i.e. as $\sim r^3$, while the viscous forces unifying the motion of particles and
 169 fluid scale with particle surface, i.e. as $\sim r^2$. These particles are illuminated in a single
 170 plane by using a solid-state pulse laser New Wave Solo using the second harmonics
 171 of wavelength $\lambda = 532 \text{ nm}$ (corresponding to green color). The single shot energy
 172 is 0.5 J and its duration is 5 ns , thus the peak power is 100 MW . The laser produces
 173 a pair of pulses separated by time-interval $10^{-5} - 10^{-4} \text{ s}$ in dependence on the flow
 174 velocity. The laser beam is defocused by using a cylindrical optics in order to illuminate
 175 the plane. The camera is oriented perpendicularly to this plane and focused into it.
 176 The Mk II Flow Sense CCD camera has resolution of 2048×2048 pixels; this camera
 177 is able to resolve both pulses of the laser into a pair of expositions. The obtained
 178 pairs of photographs of particles are processed by using the Dantec Dynamic Studio
 179 software¹. The vector fields contain 64×64 vectors, a single grid point covers area of
 180 $0.53 \times 0.53 \text{ mm}^2$ corresponding to $6.65 \cdot 10^{-3}$ times the chord length c . The analysis
 181 continues by using our custom-made software: one grid point is removed from each side
 182 getting resolution 62×62 vectors, because the vectors at the edge of the field of view are

¹ Details for readers familiar with this software: (1) first 5 frames are removed from each movie, (2) the minimum of image ensemble is calculated by using `Image Min/Max` function, (3) this minimum is subtracted from each image in the ensemble by using the `Image Arithmetic` function, (4) the velocity vectors are calculated by using the `Adaptive PIV` function with parameters: grid step size 32, minimum IA size 32, maximum IA size 32, validation based on universal outlier detection over neighborhood 5×5 , normalization 0.1, acceptance limit 2.0, the algorithm adapts to the particle density with particle detection limit of 5 to the desired number of 10 particles per IA; it adapts IA shape to velocity gradients with $|Ux|, |Vx|, |Uy|, |Vy| \leq 0.1$ and $\sqrt{Ux^2 + Vx^2 + Uy^2 + Vy^2} \leq 0.2$, the convergence limit is 0.01 pixel and the maximum number of iterations is 10. (6) The last step within the Dantec software is the export via `Numeric Export` function.

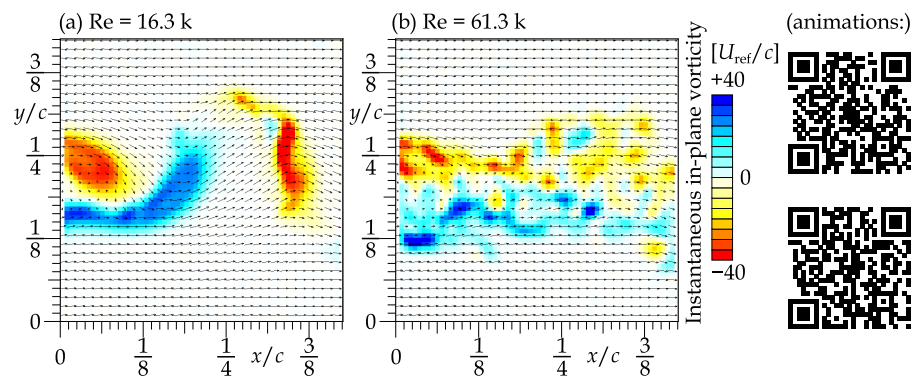


Figure 3. The example of instantaneous velocity fields at chord-based Reynolds numbers $1.63 \cdot 10^4$ (a) and $6.13 \cdot 10^4$ (b), both at zero angle of attack. Only every second velocity vector is displayed. The color in background represents z-component of vorticity $\omega = \nabla \times \vec{u}$. The QR-codes link to http://home.zcu.cz/~dudad/PIV_uplav_3ms.gif and http://home.zcu.cz/~dudad/PIV_uplav_12ms.gif, which show pair of photos used for drawing this figures.

183 not trustworthy, as there is relevant number of particles leaving or entering the field of
 184 view during the *time between laser pulses*. Second step is to filter out the snapshots, whose
 185 small-scale energy content is significantly greater than the average of the ensemble. This
 186 procedure serves for removing rough errors, whose smallest-scale energy content is
 187 typically large. This method is in more details described in our previous article [24]. In
 188 fact, current data have reasonable quality, therefore, this procedure removes at least one
 189 snapshot only for 40 % of datasets. The example of obtained instantaneous velocity field
 190 is displayed in Figure 3.

191 2.6. Lift force measurement

192 Wind-tunnel balance is designed to measure the lift force L on the airfoil. The basis
 193 of the balance is a resiliently suspended movable table. Elastic suspension created with 4
 194 parallel flexible elements. The elastic suspension has the lowest stiffness in the direction
 195 of the force L and in all other directions the stiffness is an order of magnitude higher.
 196 The displacement of the table is measured by an eddy-current sensor which responds to
 197 the distance to the steel target. Unwanted vibrations are dampened by eddy-currents
 198 in the copper sheet as it moves near to the neodymium super magnet. The output of
 199 the sensor is calibrated by a known force created by using a calibration weight in the lift
 200 direction.

201 3. Results and discussion

202 3.1. Zero angle of attack – average velocities

203 The average stream-wise velocity component for four different Reynolds numbers
 204 is displayed in Figure 5. In that figure, the spatial distribution for each case (velocity and
 205 airfoil realization) is plotted as a colormap. The direct comparison of isotachs (contour
 206 lines of velocity) for each product is plotted in a single panel in order to see the differences
 207 among those products. Figure 6 shows the profile of average stream-wise velocity at
 208 distance $10 \text{ mm} = 1/8c$ past the trailing edge. This plot contains less information than
 209 the previous one, but this information is organized in a different manner in order to
 210 catch the differences better.

211 First look to figures 5 and 6 reveals, that the largest difference occurs at Reynolds
 212 number $4.08 \cdot 10^4$ (denoted 40.8k, where k plays for $\cdot 10^3$ as a standard multiplier of
 213 SI system). This Reynolds number is close to a critical Reynolds number of transition
 214 between wake dominated by von-Kármán vortex street [25] and the wake dominated by
 215 continuous boundary layers developed along the airfoil, as will be apparent from next

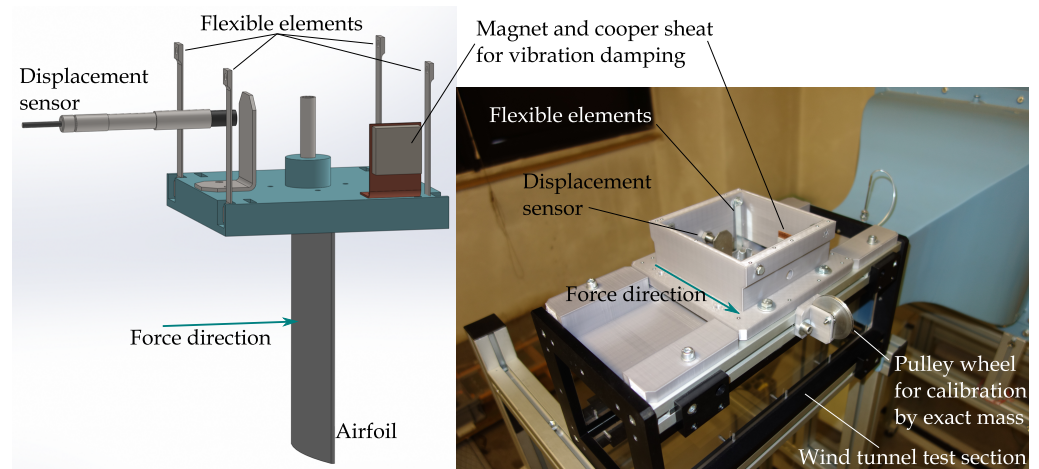


Figure 4. Sketch and photograph of the balance used for lift force measurement. The metallic flexible elements are connected to the wind tunnel walls at their upper end. The airfoil is reinforced by using an aluminum tube, which connects the airfoil to the movable table as well. The eddy-current sensor is used to electrically measure the displacement of entire table.

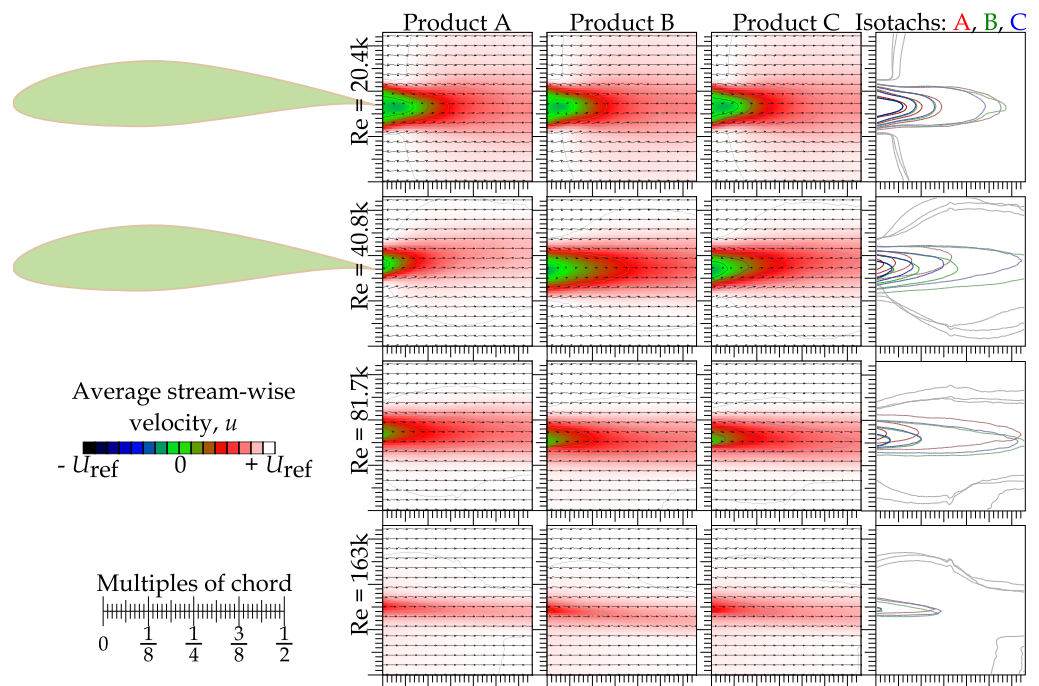


Figure 5. First three columns show the map of average stream-wise velocity u normalized by the reference velocity U_{ref} past the slightly different samples A, B and C. Fourth column displays isotachs together. The green tips in the left edge of figure represent the positions of airfoil trailing edge. Reynolds number is based on chord length and the reference velocity. The reference velocity is measured in the empty wind tunnel under otherwise similar condition. Angle of attack $\alpha = 0^\circ$ in all panels.

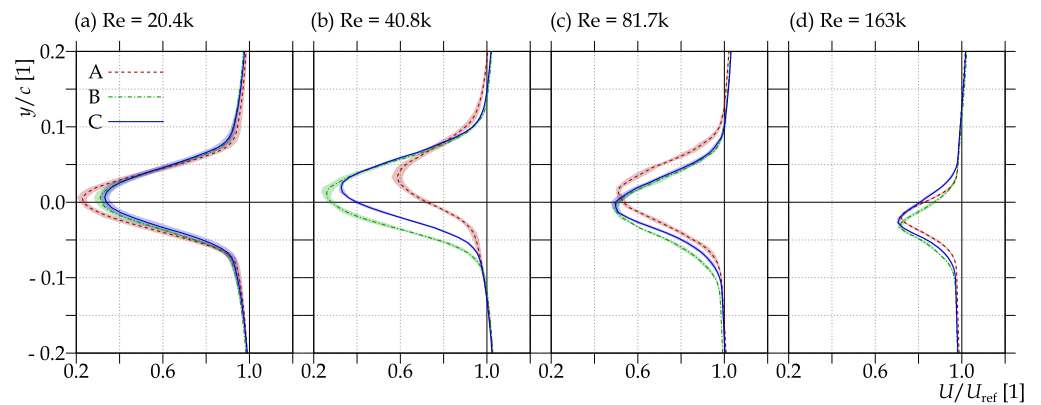


Figure 6. The cross-stream profile of average stream-wise velocity $\langle u \rangle$ at stream-wise distance $10 \text{ mm} = 1/8c$ past the trailing edge. Angle of attack $\alpha = 0^\circ$ in all panels. The uncertainty is displayed via the area.

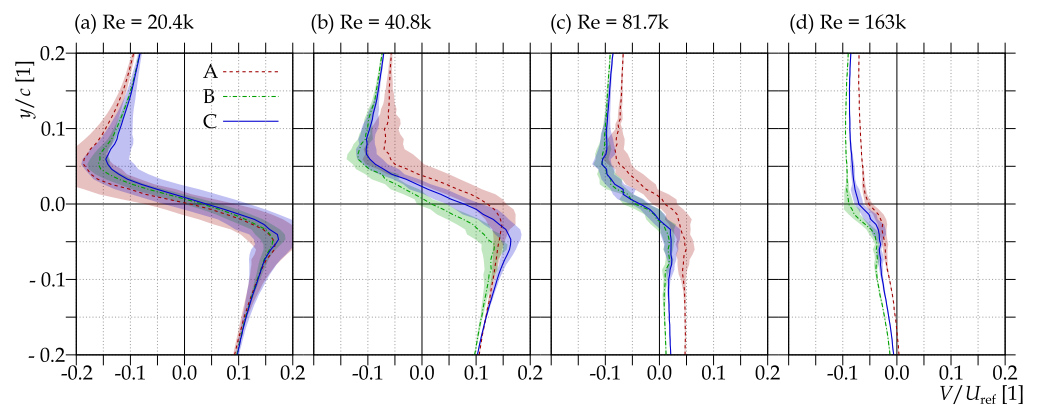


Figure 7. The cross-stream profile of average cross-stream velocity $\langle v \rangle$ at stream-wise distance $10 \text{ mm} = 1/8c$ past the trailing edge. Angle of attack $\alpha = 0^\circ$ in all panels.

216 figures analyzing turbulent kinetic energy (mainly the ratio $\sigma[v]/\sigma[u]$ in Figure 15). As
 217 it was mentioned above, the variant A is slightly smaller than B and C, which both are
 218 expanded by Minkowski sum with circle of radius 100 mm . Therefore, it is natural to
 219 expect, that the *real* Reynolds number for those variants is slightly larger than that for
 220 variant A, thus the transition occurs at slightly smaller velocities.

221 The next largest discrepancy lies in the *position* of the wake past variant A. It is best
 222 apparent at Reynolds number $8.17 \cdot 10^4$, although at others it applies as well. This shift
 223 is caused by the missing trailing edge of the variant A, see Figure 1. As the trailing edge
 224 is slanted towards the pressure side (direction down in the figures), the entire airfoil
 225 ends upper (in figures) and thus the entire wake is shifted up. One may ask, why this
 226 shift is less apparent at the lower Reynolds number? We think, that it is due to the
 227 different mechanism of wake creation – the von Kármán vortex street is formed due to
 228 Kelvin-Helmholtz instability of the shear layer between the fluid, which passes far from
 229 the obstacle, and between the fluid, which is slowed by the presence of the obstacle. But
 230 this difference starts at the *leading edge*, which is much more similar among different
 231 airfoil variants, see Figure 1(b).

232 3.2. Balance measurement of lift coefficient

233 According to the momentum balance principle, the lift force acting one direction
 234 has to deflect the fluid motion in opposite direction. However, the true integration of the
 235 momentum deflected needs the knowledge of the flow field around the entire airfoil,

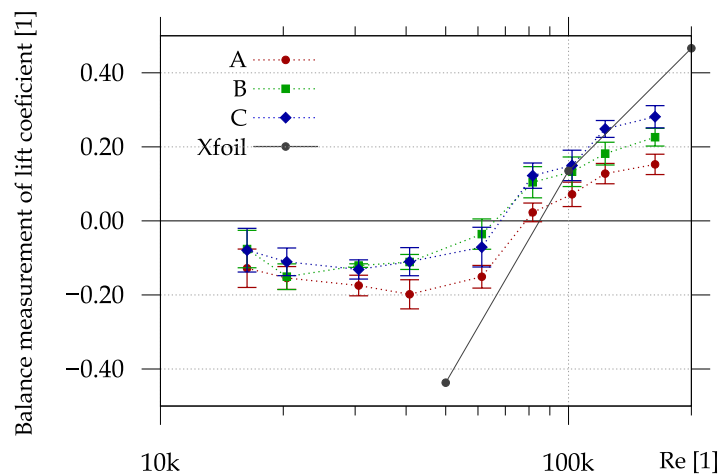


Figure 8. Balance measurement of the lift coefficient. The theoretical results for ideal geometry are displayed as well. Angle of attack $\alpha = 0^\circ$.

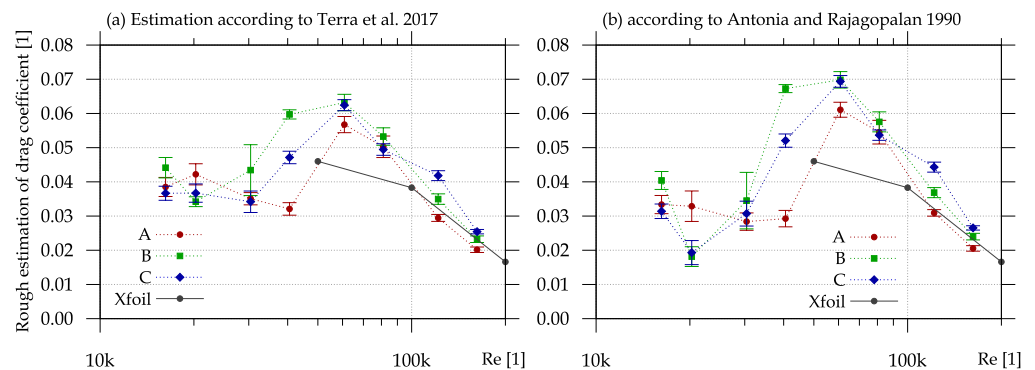


Figure 9. Rough estimation of the drag coefficient based only on the momentum deficit and stress term, the pressure term is ignored. Black points denote data obtained from public database Airfoiltools [3] calculated by using the Xfoil [5,26,27] for the $N_{crit} = 9$. Angle of attack $\alpha = 0^\circ$ in all panels.

236 which is not available. In order to get some information about the lift, which is anyway
 237 the crucial feature of any airfoil, we measured it by using a force balance equipped with
 238 four flexible metal elements in order to allow deformation in a single direction only, see
 239 figure 4. The resulting lift coefficient is shown in Figure 8 for angle of attack equal to 0°
 240 and for Reynolds numbers in the same range as used for the PIV measurement. This
 241 figure shows a reasonable match with the prediction by the Xfoil algorithm [5,26,27] at
 242 Reynolds number 10^5 , but the values are heavily different at lower and higher velocities.
 243 Differences among airfoil realizations shows, that the A variant performs lowest lift
 244 as expected. The geometrically best airfoil C has the best aerodynamic performance.
 245 Interestingly, the A airfoil has larger negative values of lift at low velocities. The large
 246 errors reported here are cause by the unwanted oscillations at higher velocities (despite
 247 the damping by using highly-conductive cooper and a strong magnet). At smaller values,
 248 the large error is caused simply be the uncertainty of the displacement sensor and of
 249 the Analog-Digital converter, because the forces are by two orders of magnitude smaller
 250 than the range adapted to fit the expected forces at largest velocities.

251 3.3. Rough estimation of drag coefficient

The time-average drag force $\langle F_D \rangle$ can be roughly estimated according to the principle of momentum balance:

$$\frac{\langle F_D \rangle}{\rho} = \int_{\text{wake}} (U_\infty - \langle u \rangle) \langle u \rangle dS - \int_{\text{wake}} \sigma^2[u] dS + \int_{\text{wake}} (p_\infty - \langle p \rangle) dS, \quad (1)$$

where U_∞ is the velocity in the unperturbed state, see the work of Terra et al. [28,29] for more details; dS is the surface element across the wake and $\sigma^2[u]$ is the variation of u . This fluctuation term originates from the averaging of Reynolds-decomposed momentum balance equation and it *decreases* the resulting force. The calculation of the last term with static pressure p is rather complicated [30] as all components of velocity gradient are needed to be known in order to integrate the pressure Poisson equation [31]. Terra et al. [28] used tomographic PIV in order to get this term and they found that, fortunately, the pressure term is important only up to the recirculation zone, it disappears downstream, see Figure 13 in [28]. This observation allows us to ignore this term and estimate the drag force only by using the measured spatial velocity distribution as

$$\frac{\langle F_D(x) \rangle}{\rho H} \approx \int_{\text{FoV}} (U_0 - \langle u(x) \rangle) \langle u(x) \rangle - \sigma^2[u(x)] \, dy, \quad (2)$$

252 where the integration is performed along y -direction in the measured two-dimensional
 253 FoV at fixed x -position, which is far enough to ignore the pressure term. In our case
 254 $x = 0.4c$ is chosen (i.e. the most downstream position within our FoV). H is the height
 255 of prismatic airfoil. The background velocity U_∞ is replaced by the background U_0 of
 256 Gaussian fit of the velocity profile (equation (5)).

There exist another approach to estimate the drag coefficient by using the spatial distribution of measured velocity across the wake. Antonia and Rajagopalan [32] offer an alternative procedure including the transverse velocity fluctuations as well

$$\frac{\langle F_D \rangle}{\rho H} \approx \int_{\text{FoV}} (U_0 - \langle u \rangle) \langle u \rangle + \sigma^2[u] - \sigma^2[v] \, dy. \quad (3)$$

257 The fluctuations in stream-wise direction are though to increase the drag force, while in
 258 Terra's definition (2), they decrease the force. At higher velocities, the estimations are
 259 very similar both dominated by the first term of the average momentum deficit; at middle
 260 Reynolds numbers, the stream-wise fluctuations are more important and the Antonia's
 261 approach gives larger values than the Terra's one. At low Reynolds numbers, the wake
 262 is dominated by transverse fluctuations associated with the vortex street decreasing the
 263 drag, which is revealed when using the Antonia's formula. The later formula is widely
 264 used in literature; e.g. Zhou et al. [33] compared the direct force measurement by using
 265 load-cell with this formula applied to the transverse profiles measured by using Laser
 266 Doppler Anemometry (LDA) technique. Mohebi et al [34] uses this formula to study the
 267 flow past a flat plate at high angles of attack.

268 3.4. The wake width and centerline

By using the spatial distribution of average stream-wise velocity, the wake centerline can be determined. It is a set of points, where the stream-wise velocity component is minimal, i.e. the velocity deficit $u_d = U_{\text{ref}} - \langle u \rangle$ is maximal. The maximum or minimum are by zero derivation:

$$\frac{d\langle u \rangle}{dy} = 0 \quad (4)$$

269 The respective isolines for different Reynolds numbers are shown in Figure 10. The main
 270 line along the axis shows the wake centerline, where the average stream-wise velocity
 271 is minimal. However, there are visible another structures as well: at low Re , there are

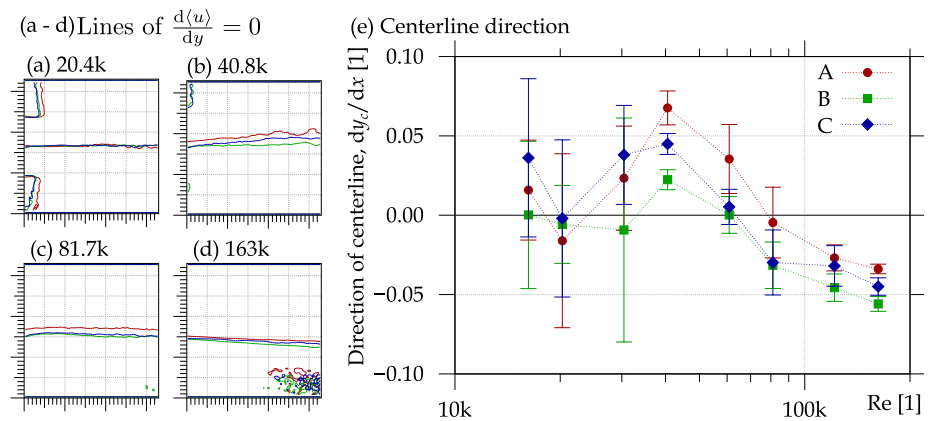


Figure 10. (a – d) Lines of extrema of $\langle u \rangle$ along cross-stream direction (y). The number in top right corner of each panel is chord-based Reynolds number and k denotes $\cdot 10^3$. (e) Direction of centerline as a function of Reynolds number. It is obtained as a linear fit of the coordinates of minima of transverse profiles. Angle of attack $\alpha = 0^\circ$ in all panels.

272 depicted the maxima caused by the acceleration along the airfoil enforced by the mass
 273 balance. At highest Re , some noise appears in the bottom right corner.

The wake width δ_w can be calculated by using multiple approaches [35] in dependence which physical property has to be explored. [35] shows that for comparison purposes the most suitable wake width is determined as twice the σ -parameter of a Gaussian function fitted to the ensemble-average stream-wise velocity profile:

$$u(y) \approx G(y) = U_0 - u_d e^{-\frac{(y-y_c)^2}{2\sigma^2}}, \quad (5)$$

274 where U_0 plays a role of a background, u_d is the deficit velocity, y_c is the y -coordinate of
 275 centerline and σ can be interpreted as half of the wake width. The Gaussian function fits
 276 well, as suggested e.g. by [36] (where the wake past circular cylinder is explored by using
 277 Acoustic Doppler Velocimetry.) The dependence of $\delta_w = 2\sigma$ on the stream-wise distance
 278 from the airfoil trailing edge is displayed in Figure 11 together with the deficit velocity
 279 u_d for each stream-wise distance x . We see that the wake width increases with distance,
 280 however, at small Reynolds number, there is a minimum of the wake width. This
 281 minimum would be more apparent when taking into account the fluctuations as shown
 282 in [35]. The observed growth of wake width is approximately linear, but, according to
 283 Eames [36], the transition to $\sim x^{0.5}$ growth can be expected downstream. The power
 284 of wake width growth rate is locked with the power of velocity deficit decrease by the
 285 conservation of momentum deficit within the wake. However, this rule do not need to
 286 be valid in the near wake, where the pressure field plays its own role.

287 The evolution of δ_w with Re at fixed distance $x = 0.4c$ is plotted in Figure 12 as well
 288 as the wake growth rate a and the velocity deficit at fixed distance.

289 3.5. Turbulent kinetic energy

The turbulent kinetic energy (TKE) is calculated as

$$E_T(\vec{x}) = \frac{1}{2} \left\langle u^2(\vec{x}) - \langle u(\vec{x}) \rangle^2 + v^2(\vec{x}) - \langle v(\vec{x}) \rangle^2 \right\rangle, \quad (6)$$

290 where u and v are the stream-wise and cross-stream velocity components measured by
 291 using PIV, \vec{x} is the position within the field of view and the sharp brackets $\langle \cdot \rangle$ denote
 292 the averaging over the ensemble. Note that this definition of TKE contains the in-plane
 293 velocity components only, therefore its value is roughly underestimated by the entire z -
 294 component term. The level of underestimation may vary with the regime of flow past the

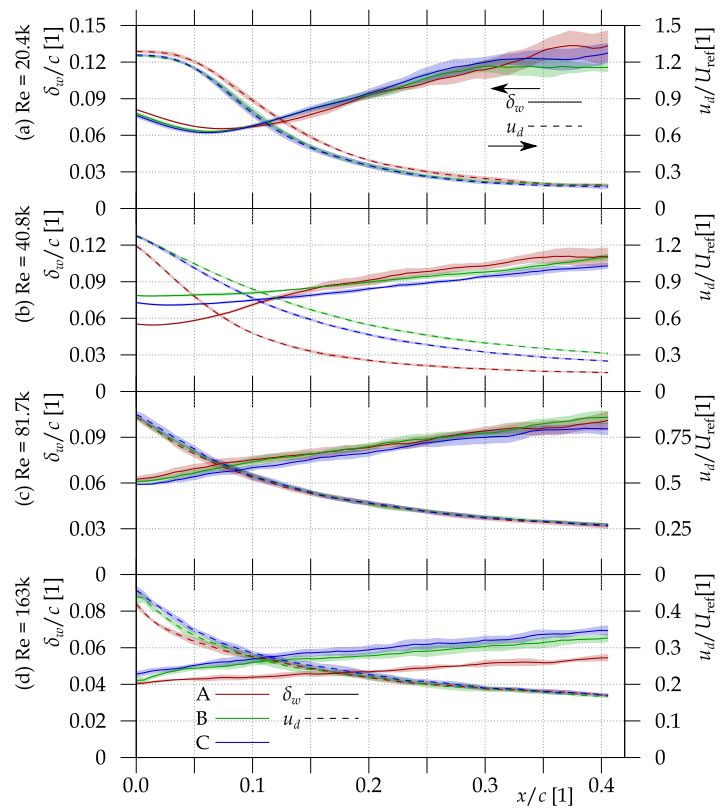


Figure 11. Solid lines indicate the wake width as a function of distance past the airfoil trailing edge, x ; the corresponding axis is on the left-hand-side of the plots. Dashed lines represent the maximum deficit velocity as a function of x ; the axes for these values are on the right-hand-side of the plots. Different panels contain data at different Reynolds numbers and angle of attack $\alpha = 0^\circ$.

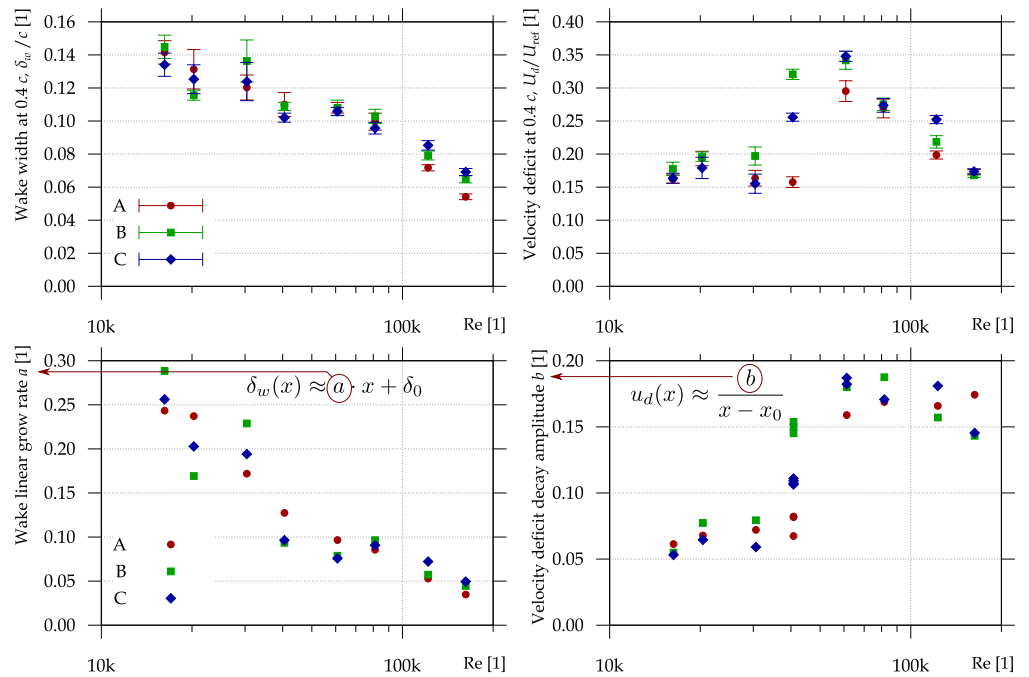


Figure 12. (Top left) Dependence of the wake width δ_w at fixed stream-wise distance $x = 0.4c$ past the trailing edge on the chord-based Reynolds number. (Top right) Dependence of the velocity deficit at certain distance $x = 0.4c$ on the Reynolds number. (Bottom left) Dependence of the grow rate of the wake width a on the Reynolds number. (Bottom right) decay rate of velocity deficit, if a hyperbolic decrease is expected. Angle of attack $\alpha = 0^\circ$ in all panels.

295 airfoil: at the smaller Reynolds number, the turbulent structures are large and oriented
 296 with the height of the airfoil, at large Reynolds numbers, the turbulent structures might
 297 be small and isotropically oriented, thus the planar definition of TKE would contain
 298 approximately 2/3 of the true TKE. Another important note is, that the velocity $\vec{u}(\vec{x})$
 299 is already averaged over the interrogation area – it is not a true single-point quantity.
 300 Therefore, the entire amount of TKE of length-scales smaller than the interrogation area
 301 is missing.

302 Figure 13 shows the spatial distribution of TKE based on two measured velocity
 303 components. The first impression is the decrease of TKE/U_{ref}^2 with Reynolds number
 304 (note the colorscale of the last row of Figure 13 is multiplied by 8). The worst product A
 305 creates significantly smaller amount of TKE, mainly at largest Re . An regime switch can
 306 be observed at Reynolds number $4.08 \cdot 10^4$, where the variant A lies in the previous stage
 307 with a single massive spot of TKE, while B and C wakes consist of a pair of maxima past
 308 the boundary layers.

309 This change of regime is even better apparent in the plot of $\sigma[v]/\sigma[u]$ in Figure 15.
 310 In respect to the original work of Romano [37], the logarithm is used in order to make
 311 the anisotropy symmetrical, because the logarithm maps the interval $(0, \infty)$ with center
 312 in 1 to interval $(-\infty, \infty)$ with center in 0. Thus the areas, where fluctuations in both
 313 investigated directions are statistically same, have got value around 0 plotted as white
 314 color; the areas, where the fluctuations in cross-stream directions are twice stronger than
 315 in the stream-wise direction, have got the value +1 and they are depicted by a blue color.

316 At smaller Re , the strong dominance of fluctuations in cross-stream direction is
 317 observed. This is caused by the von Kármán vortex street formed past the airfoil. The
 318 areas at the left-hand-side of the FoV, but not behind the airfoil, where the stream-wise
 319 fluctuations dominate, can be caused by the unsteadiness of the wind tunnel velocity,
 320 which is a known issue at smaller velocities. This fluctuations occur in the stream-wise
 321 direction only.

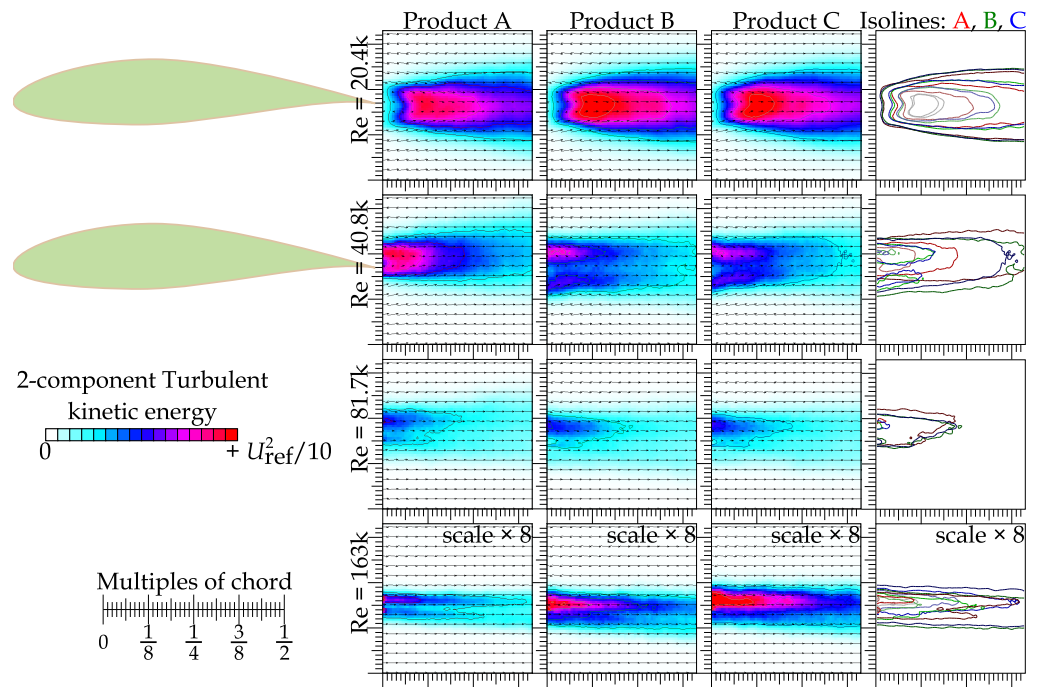


Figure 13. Turbulent kinetic energy (TKE) based on in-plane velocities only (thus it is thought to be underestimated) obtained behind three different manufactured samples denoted A, B and C. The scale is adapted to $\frac{1}{10} U_{\text{ref}}^2$ except for the last line, where the scale is $\frac{1}{80} U_{\text{ref}}^2$. Angle of attack $\alpha = 0^\circ$ in all panels.

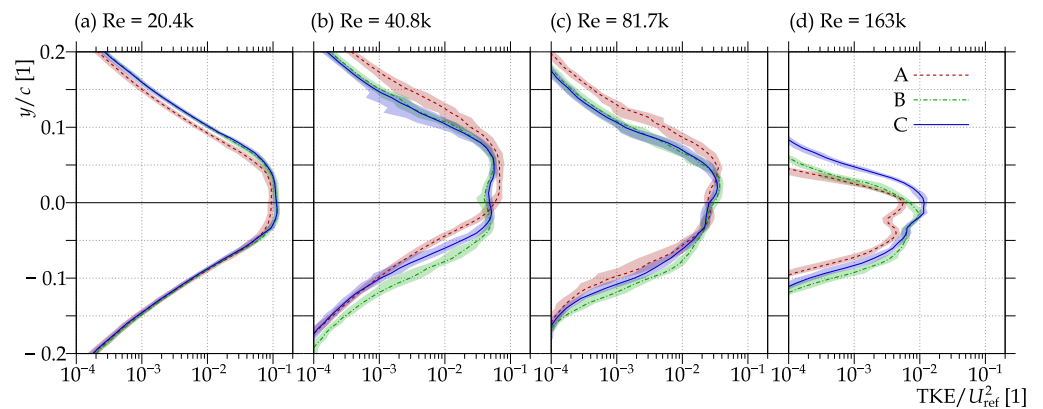


Figure 14. The cross-stream profile of the turbulent kinetic energy (TKE) at stream-wise distance $10 \text{ mm} = 1/8c$ past the trailing edge. Angle of attack $\alpha = 0^\circ$ in all panels. Note the logarithmic scale!

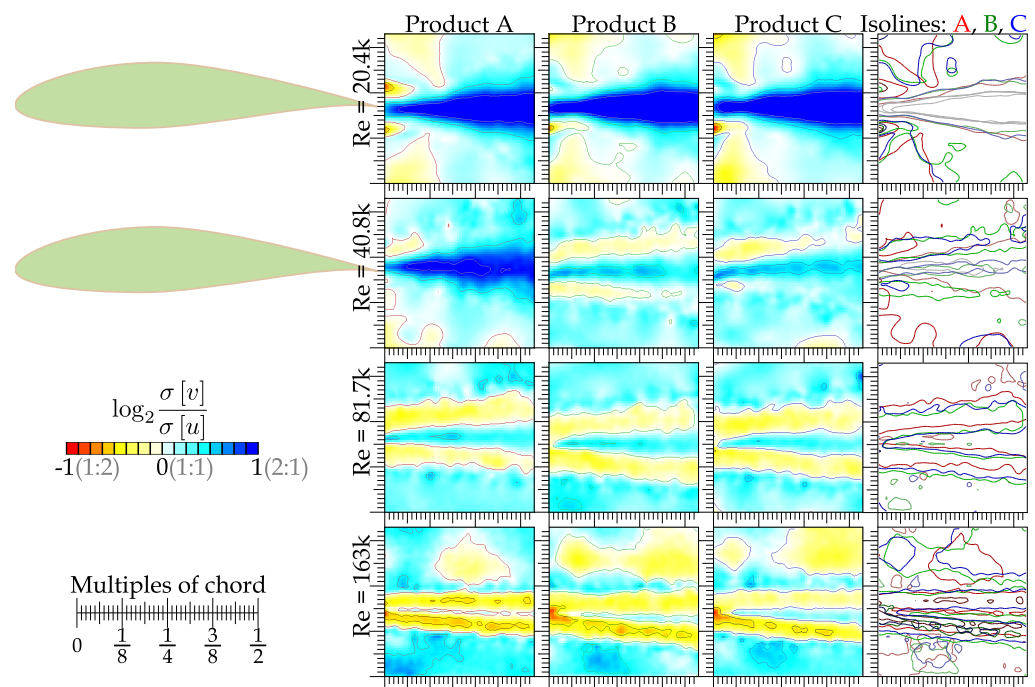


Figure 15. Panels (a - c) show the ratio of cross-stream fluctuations to stream-wise fluctuations obtained behind three different manufactured samples denoted A, B and C. Angle of attack $\alpha = 0^\circ$ in all panels. Redder colors signify dominance of stream-wise fluctuations, which is typical for areas of continuing boundary layer, while the bluer colors highlight areas of stronger cross-stream fluctuations, which is typical for the so-called *true wake*.

322 At $Re = 4.04 \cdot 10^4$, the regime changes in favor to the stream-wise fluctuations.
 323 However, there is a central strip of cross-stream fluctuations dominance, which is con-
 324 nected with the von Kármán vortex street, which is still present at those velocities. The
 325 central blue strip weakens with increasing Reynolds number, but it still exists at the
 326 largest explored velocity.

327 The dominance of stream-wise fluctuations is caused by the continuation of bound-
 328 ary layers formed along the airfoil. The length-scale of a fluctuation inside bound-
 329 ary layer is limited in cross-stream direction by the thickness of the boundary layer, while the
 330 stream-wise fluctuations are not limited in their development. The level of anisotropy
 331 becomes stronger at the pressure side of the airfoil (bottom part of the figures) and at the
 332 largest explored Re.

333 The classical approach of investigating anisotropy [38] uses all three instantaneous
 334 velocity components measured at least in single point by 3-wire Hot Wire Anemometer
 335 [39] or by using Stereo PIV [40–42]. In this experiment, we have only two velocity
 336 components, although Stereo-PIV measurement is *planned to the future*. In the case of 2D
 337 velocity components, the eigen values cannot be extracted, thus we are limited to the
 338 ratio of fluctuations in two directions, which is already used in article by Romano [37].
 339 Another approach is to use the so-called *degree of anisotropy*, where the difference plays
 340 a role instead of the ratio, as it is used e.g. in the book [43]. The spatial distributions
 341 obtained by using the *degree of anisotropy* or by using the ratio of $\sigma[v]/\sigma[u]$ are very
 342 similar.

343 3.6. TKE by length-scale of fluctuations

A better insight into the nature of turbulent kinetic energy can be achieved by
 looking, which are the sizes of fluctuations producing the TKE [1]. This can be done by
 using our approach [24,44] of separating the fluctuations by length-scale inspired by the
 work of Agrawal and Prasad [45,46]. The spatial spectrum [24] is obtained without a

need of temporal resolution [47]. The idea is quite simple: spatially resolved velocity field $\vec{u}(\vec{x})$ is convoluted with a band-pass-filter:

$$\vec{u}_{lh}(\vec{x}) = \vec{u}(\vec{x}) * P_{lh}(\vec{x}), \quad (7)$$

where $P_{lh}(\vec{x})$ is the band-pass-filter keeping structures of size larger than σ_l and smaller than σ_h . It is obtained as a difference of two Gauss functions:

$$P_{lh}(\vec{x}) = \frac{1}{2\pi\sigma_l^2} e^{-\frac{x^2}{2\sigma_l^2}} - \frac{1}{2\pi\sigma_h^2} e^{-\frac{x^2}{2\sigma_h^2}}. \quad (8)$$

Then, the spatial distribution of turbulent kinetic energies of different bands $\vec{u}_{lh}(\vec{x})$ is calculated. Figure 16 displays combination of spatial distributions of TKE of three bands depicted by three basic colors: red for the TKE of spatially smallest fluctuations of length interval 1.0 – 1.5 grid points, which corresponds to $7 \cdot 10^{-3} - 1 \cdot 10^{-2}c$, c chord width. The green channel shows fluctuations of length interval 3 – 4 grid points, i.e. $(2.0 - 2.7) \cdot 10^{-2}c$ and the blue colors represent the largest resolved fluctuations of sizes between 8 – 12 grid points, i.e. $(5.3 - 8.0) \cdot 10^{-2}c$. The relative intensities $\hat{I}(k)$ of each color channel have to be normalized according to the band interval $k/\Delta k^2$. Additionally, it is normalized by the famous Kolmogorov scaling [48] $k^{-5/3}$ in order to get equal color intensities in the case, when the power spectral density follows this scaling;

$$\hat{I}(k) \sim E_T(k) \cdot \frac{k}{\Delta k^2} \cdot k^{\frac{5}{3}}, \quad (9)$$

where the effective wave number of an explored band of fluctuations with the size between σ_l and σ_h is calculated as

$$k = \frac{2}{\sigma_l + \sigma_h} \quad (10)$$

and

$$\Delta k = \frac{1}{\sigma_l} - \frac{1}{\sigma_h}. \quad (11)$$

344 By using more bands within the spatial resolution of our data, the spatial power spectral
 345 density $E(k)/k$ can be reconstructed. It is important to note, that in respect to the classical
 346 spectra obtained from time-resolved point data (e.g. [49–51]), our method is limited
 347 to the range of field of view to the grid point. This interval covers only values from
 348 0.53 mm to 32 mm, i.e. one and half orders of magnitude. The classical time resolved
 349 point measurements typically cover tens of minutes by resolution of tens of kilohertz
 350 covering five orders of magnitude in frequency [52]. Our approach shows the spectra
 351 calculated by using the entire field of view, each point is affected by all other points in
 352 the data ensemble. Moreover, the signal by larger scales feels larger neighboring, while
 353 the smallest scale signal is almost local.

354 The spatial distribution of length-scale dependent TKE is shown in Figure 16. At
 355 lower Re , there are large areas displaying relative large-scale source of TKE, which is
 356 displayed as a massive blue spot. This is caused by large almost laminar vortices in
 357 the forming von Kármán vortex street. At $Re 4.08 \cdot 10^4$, the wake past the product A
 358 belongs rather to the previous regime, while the others to the next regime characterized
 359 by continuing boundary layers containing small scale fluctuations. The wake is still
 360 affected by larger scale oscillations of the wake, thus it is displayed in violet colors – it
 361 contains large scales and small scales, the middle scales are depleted. At even higher
 362 Re , the importance of large-scales decreases, and the asymmetry appears between the
 363 pressure and suction side of the wake – there is more middle scale fluctuations (green)
 364 past the pressure side. At the highest explored $Re 1.63 \cdot 10^5$, the large-scales are weaker,
 365 the wake consists of a pair of strips of middle- and small-scale fluctuations (yellow)

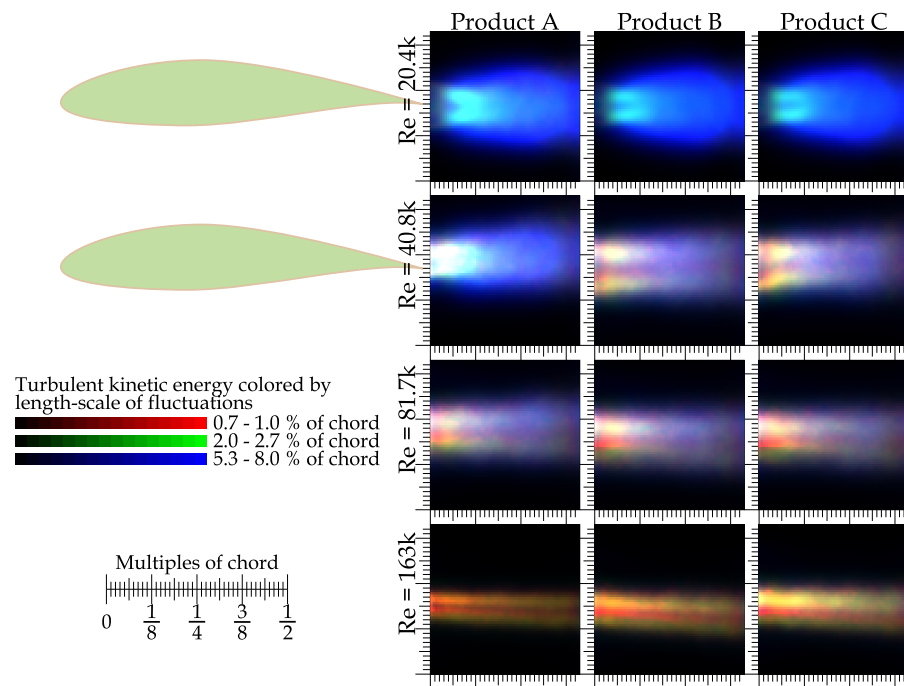


Figure 16. Turbulent kinetic energy colored by the length-scale of fluctuations producing it at angle of attack $\alpha = 0^\circ$. TKE is divided into three channels: red channel represents fluctuations of size 0.7 – 1.0% of chord width, green channel shows fluctuations of size between 2.0% and 2.7% of chord width, while the blue plays for the largest fluctuations of size 5.3 – 8.0% of chord. The colorscale among different color channels is normalized in such a way, that an ideal Kolmogorov turbulence would be displayed in shades of gray. Among different Re and variants, the colorscale is automatically adapted (for differences in amount of TKE look to Figure 13 or Figure 14).

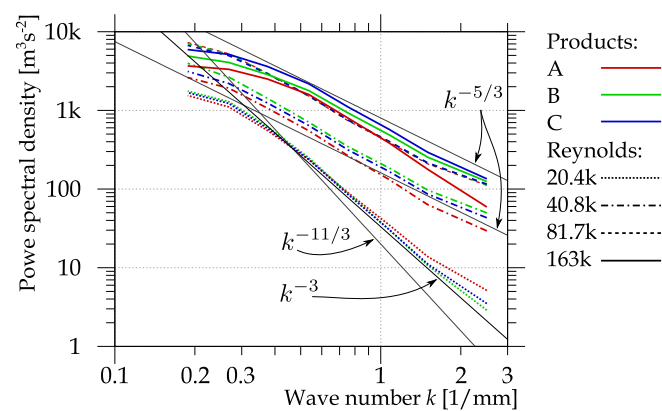


Figure 17. Spatial spectra of turbulent kinetic energy. The different products are distinguished via color, the Reynolds number via line style angle of attack is zero in all cases. Thin lines represent scalings: $k^{-5/3}$, k^{-3} and $k^{-11/3}$.

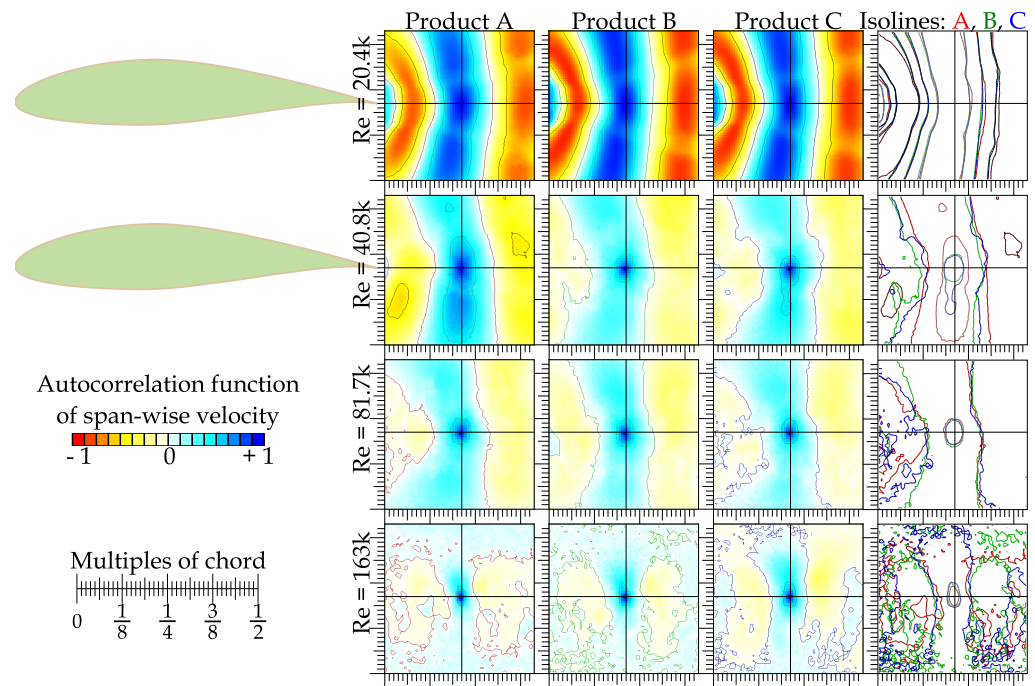


Figure 18. Autocorrelation function of cross-stream velocity component v with a reference point in the middle of the field of view, i.e. $0.21c$ past the trailing edge at $\alpha = 0^\circ$.

366 with a strip of small scales (red) in between. Lets mention again, that the dominance
 367 of some color in Figure 16 signifies, that there is slightly more energy in corresponding
 368 length-scale-band, than it would be in the case of ideal turbulence following $5/3$ law.
 369 Therefore, the energy content of large-scale fluctuations at the highest Reynolds number
 370 is still larger than the two other band combined in absolute numbers. This issue is better
 371 apparent in the plot of power spectral density in Figure 17.

372 The power spectral density, Figure 17, displays k^{-3} scaling at the smallest Re. Lets
 373 mention, that even steeper scaling $k^{-11/3}$ is typical for two dimensional arrangement of
 374 vortices [53], which is typically observed in low-Reynolds number wake past circular
 375 cylinder. At higher Reynolds numbers, the spectra approaches the $k^{-5/3}$ law. Note, that
 376 the vertical axis is not normalized by U_{ref}^2 ; it displays absolute numbers. Despite this,
 377 the wake at $\text{Re } 1.63 \cdot 10^5$ displays less large-scale energy than wake at $\text{Re } 8.2 \cdot 10^4$, while
 378 at middle- and small-scale wave numbers, the larger Re contains more energy.

379 3.7. Spatial correlation

Another way of exploring the size of turbulent structures is the usage of the autocorrelation functions or the structure functions [54,55]. Here, the autocorrelation function of cross-stream velocity component is used, because this component is active in the formation of von Kármán vortex street observed at the lower Reynolds numbers. The autocorrelation of some quantity between two separate points \vec{x} and $\vec{\xi}$ is defined as

$$R_{vv}(\vec{x}; \vec{\xi}) = \frac{\langle v'(\vec{x}) \cdot v'(\vec{\xi}) \rangle}{\sigma[v(\vec{x})] \cdot \sigma[v(\vec{\xi})]}, \quad (12)$$

380 where $\langle \cdot \rangle$ signifies the ensemble averaging and $\sigma[\cdot]$ is the standard deviation; v' is a
 381 fluctuation velocity, $v' = v - \langle v \rangle$. In Figure 18, the position vector \vec{x} runs over the entire
 382 FoV, while the reference point $\vec{\xi}$ is fixed in the FoV center.

383 One of the natural properties of correlation function is, that it inherits some aspects
 384 of the basic function, e.g. its periodicity. Therefore, the periodic von Kármán vortex

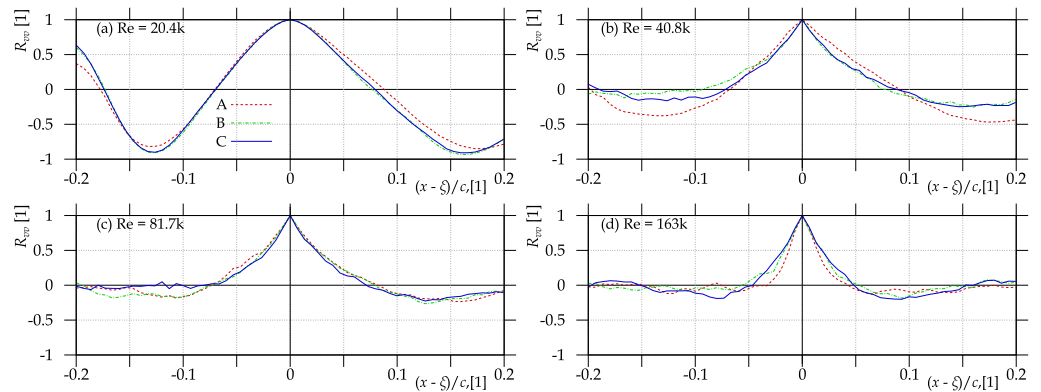


Figure 19. The stream-wise profile of the autocorrelation function of cross-stream velocity component at $\alpha = 0^\circ$.

385 street is reflected as a periodic function of autocorrelation with minima in the distance
 386 of half of the spatial period. Note that the distance of minimum is shorter in upstream
 387 direction and longer in downstream direction (e.g. at $Re = 1.63 \cdot 10^4$, Figure 19(a), the
 388 minimum in upstream direction occurs distance at $-0.15c$ for the case A and at $-0.13c$
 389 for B and C, while in the upstream direction, the first minimum is $0.18c$ for A and $0.17c$
 390 for B and C). With increasing Re , the dominant periodic behavior disappears and the
 391 R_{vv} starts to represent the spatial decay of coherence.

The level of spatial coherence is characterized by the integral length-scale, which is defined as the integral of R_{vv} up to some distance M , where R_{vv} reaches zero, $R_{vv}(M) = 0$

$$L_{vv} = \int_0^M R_{vv}(x - \xi) dx. \quad (13)$$

392 But reaching zero is not straight-forward as all experimental data contain noise. Thus
 393 the autocorrelation oscillates around zero, see Figure 19 later panels. Azevedo et al. [56]
 394 explores several possibilities of limits of this integral. Here, the simplest key is chosen:
 395 the integration is stopped in the distance, where R_{vv} reaches value $1/e \approx 0.367$. This
 396 choice underestimates the integral length-scale, as it is shown in [56], but it does it in a
 397 most systematic way.

398 The values of integral length-scales integrated up to the distance, where $R_{vv}(x) =$
 399 $1/e$, are plotted in Figure 20. The panel (a) of Figure 20 shows L_{vv} along the stream-wise
 400 direction in millimeters. L_{vv} decays systematically and continuously with increasing
 401 Re . Note that the interpretation of L_{vv} depends on the regime: at the periodic regime,
 402 where R_{vv} is a periodic function with the period of Strouhal shedding, it might represent
 403 just a constant fraction of spatial Strouhal period². At regime dominated by continuing
 404 boundary layers, L_{vv} might represent the statistical size of coherent structures, which is
 405 the original motivation for this quantity. But, a systematic shortening of L_{vv} is observed
 406 without some evidence of regime change. However, if L_{vv} is calculated along the
 407 cross-stream direction (panel (c) in Figure 20), the switch of regimes is evident. At
 408 lower Reynolds number, the observed L_{vv} along cross-stream direction is as large as the
 409 measured area. When the large von Kármán street disappears, the value of L_{vv} along
 410 cross-stream direction converges to the range observed along stream-wise direction
 411 (panel (a) of Figure 20). The lost of large-scale vertical correlation appears at the already
 412 discussed regime change at $Re = 4.04 \cdot 10^4$.

413 The autocorrelation function R_{vv} is not symmetric between upstream and down-
 414 stream direction, as it is shown in the panel (b) of Figure 20. The asymmetry reaches

² suppose that $R_{vv} = \cos ax$, where a is 2π times the spatial Strouhal frequency, then $L_{vv} = [a^{-1} \sin x]_0^M$, where $M, R_{vv}(M) = e^{-1}$, thus $M = a^{-1} \arccos e^{-1}$. Rewrite $\sin x = \sqrt{1 - \cos^2 x}$, then $L_{vv} = a^{-1} \sin \arccos e^{-1} = a^{-1} \sqrt{1 - e^{-2}} = 0.923a^{-1}$.

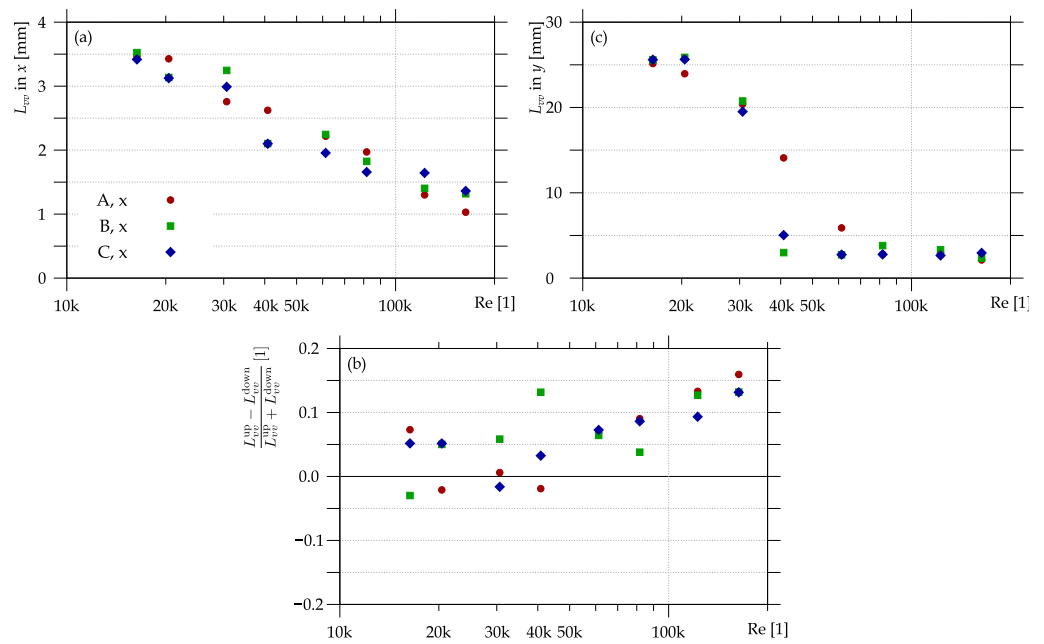


Figure 20. Integral length-scale of cross-stream velocity component v . Panel (a) shows the integral length-scale along stream-wise axis, however it is different in up-stream and down-stream direction, therefore panel (b) shows this asymmetry. Panel (c) shows the integral length-scale of v along cross-stream axis. Angle of attack $\alpha = 0^\circ$ in all panels.

415 the values of 15% in favor of the downstream direction. This effect is caused by the
 416 increasing average velocity along the stream, as the wake widens.

417 4. Conclusion

418 We asked, how the manufacturing inaccuracies affect the flow topology. The wake
 419 behind a single prismatic airfoil NACA 64(3)-618 has been studied experimentally by
 420 using the Particle Image Velocimetry (PIV) method. The airfoil has been realized in
 421 three copies of different similarity to the ideal model. The real geometry has been
 422 measured by using 3D optical scanner GOM Atos. We focused to the zero angle of attack
 423 at chord-based Reynolds number ranging from $1.6 \cdot 10^4$ to $1.6 \cdot 10^5$.

424 Generally speaking, the effect of model quality is weaker than the effect of Reynolds
 425 number – the average flow pattern differs more among different velocities than among
 426 different models. The deviations in model shape shifts the transition between turbulent
 427 regimes, which is best observable at Reynolds number $4.1 \cdot 10^4$, where the variant
 428 denoted A displays same regime as for smaller velocities, while other two realizations B
 429 and C produces wake similar to that observable at higher velocities. This regime change
 430 is apparent in average flow topology (Figure 6), in the decrease of average velocity
 431 deficit (Figure 12), in the spatial distribution of turbulent kinetic energy (Figure 13), in
 432 the analysis of isotropy (Figure 15) or in the length-scale of structures producing TKE
 433 (Figure 16). This issue can be important, if the flow machine (e.g. wind turbine) was
 434 designed to operate at edge of some regime.

435 The worst variant A differs from the ideal geometry mainly in the quality of trailing
 436 edge: it is shorter by 2.8 mm (3.5% of the chord length), it lacks the entire small trail
 437 pointing to the pressure side of the airfoil, see Figure 1. Thus it produces smallest lift
 438 (Figure 8), but also the smallest drag (Figure 9), as the other variants are slightly ex-
 439 panded by using Minkowski sum. Despite this huge discrepancy, the flow characteristics
 440 of this realization are *comparable* to the better models. The wake centerline orientation is
 441 similar to others (Figure 10), the wake thickness reaches similar values (Figure 12), and

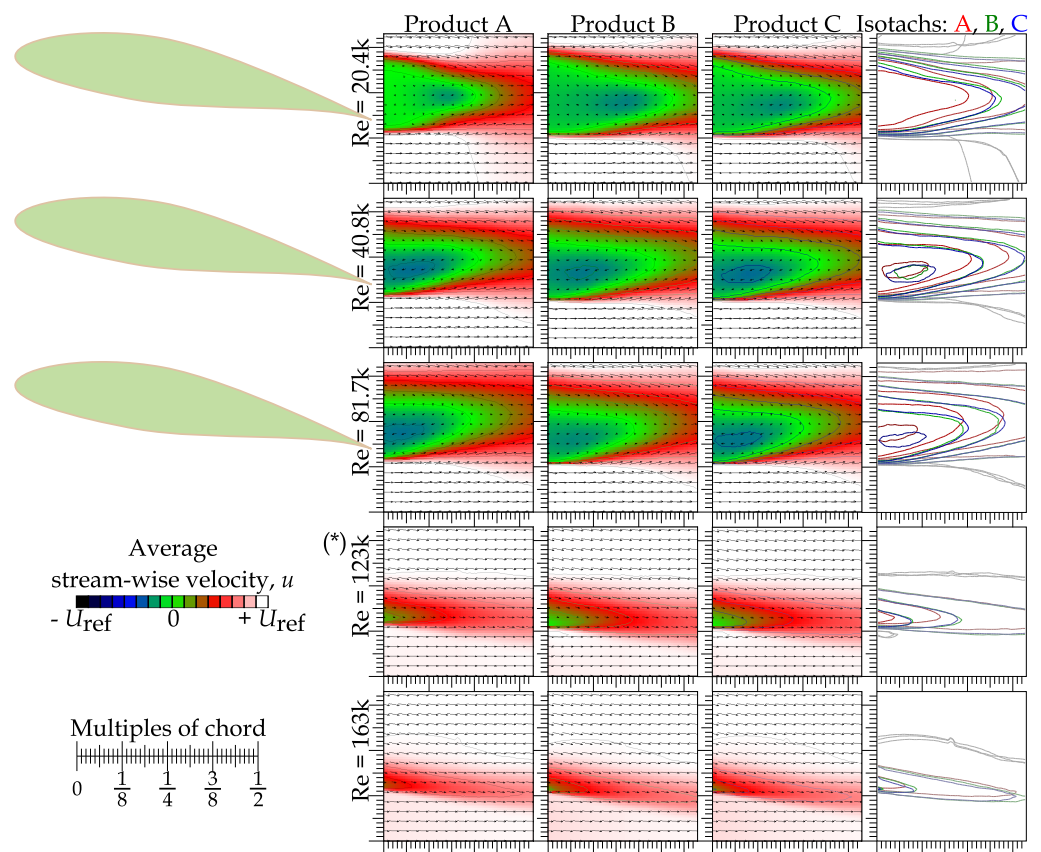


Figure A21. Map of ensemble average of stream-wise velocity u at angle of attack $\alpha = 10^\circ$. In respect to Figure 5, here is an added row denoted (*) containing data at $Re = 1.23 \cdot 10^5$ in order to show that the flow is adhered at this velocity.

442 even the length-scales of fluctuations in the wake display similar pattern (Figure 16) and
 443 spectrum (Figure 17) as better brothers.

444 **In the future, we plan to compare these results with numerical simulation covering**
 445 **not only the scanned geometry, but also the ideal design, which is not accessible by the**
 446 **experiment.**

447 Appendix E Large angle of attack

448 Increasing the angle of attack can, generally, increase the lift coefficient even at
 449 symmetrical airfoils. But it works only up to a critical angle, where the stall occurs.
 450 In this study, we measured only at angles 0° and $+10^\circ$, which is quite a large angle.
 451 The separation bubble is well resolvable in the plots of ensemble average stream-wise
 452 velocity in Figure A21. The critical velocity of stall has been not measured at certain α ,
 453 neither the critical angles of stall were explored at certain velocity. The only information,
 454 which can be read out of our data is, that at $\alpha = +10^\circ$ the stall critical Reynolds number
 455 lies between $8.17 \cdot 10^4$ and $1.23 \cdot 10^5$.

456 Similarly, the wake centerline points towards the pressure side of the airfoil since
 457 $Re = 1.23 \cdot 10^4$, at lower velocities, it points towards the suction side (up in images). The
 458 centerline is determined as set of points, where $d\langle u \rangle / dy = 0$, which highlights minima
 459 as well as maxima of the cross-stream velocity profiles, see Figure A22.

460 The difference among airfoil variants is best visible in the plot of TKE in Figure
 461 A23. The flow past variant A at Reynolds number $2.04 \cdot 10^4$ displays clearly a different
 462 regime than the flow past B and C variants. As discussed above, this discrepancy could
 463 be caused by the slightly smaller size of A and thus slightly smaller *true* Re . Thus in
 464 Figs. A23 and A24, one row is plotted in advance; that row contains data at even smaller

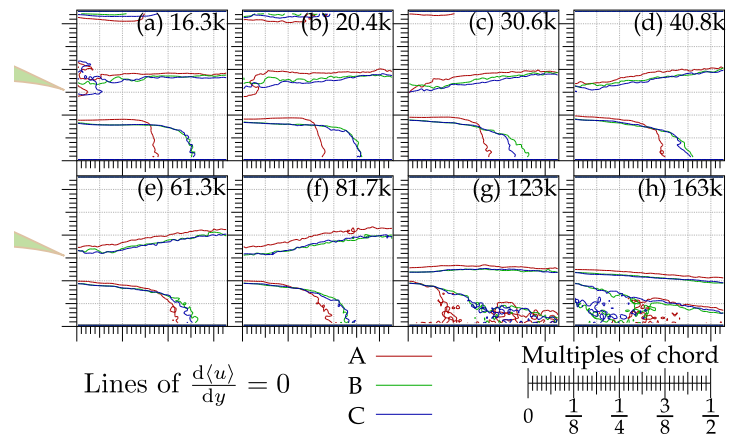


Figure A22. Lines of extrema of $\langle u \rangle$ along cross-stream direction (y). The number in top right corner of each panel is chord-based Reynolds number and k denotes $\cdot 10^3$. Angle of attack $\alpha = 0^\circ$ in all panels.

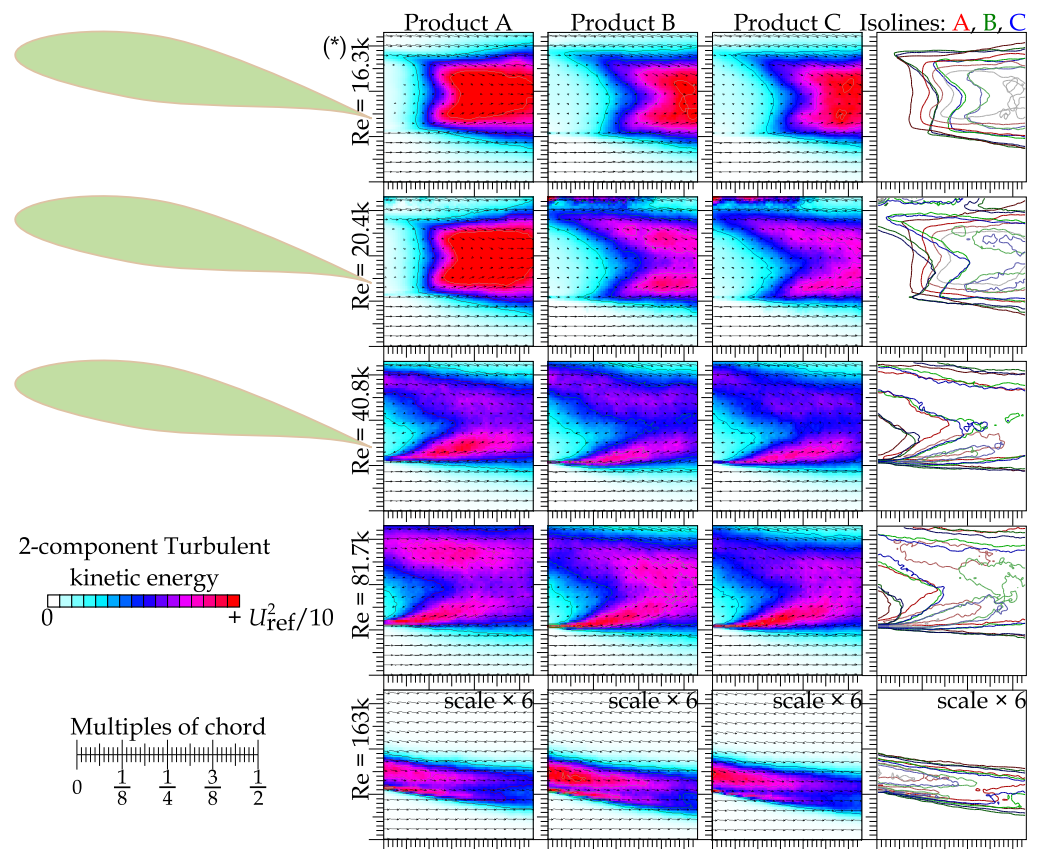


Figure A23. Map of turbulent kinetic energy at angle of attack $\alpha = 10^\circ$. The first row is added, it is denoted (*) and it contains data at $Re = 1.63 \cdot 10^4$ in order to show that the wake past product A at $Re = 2.04 \cdot 10^4$ belongs to previous regime.

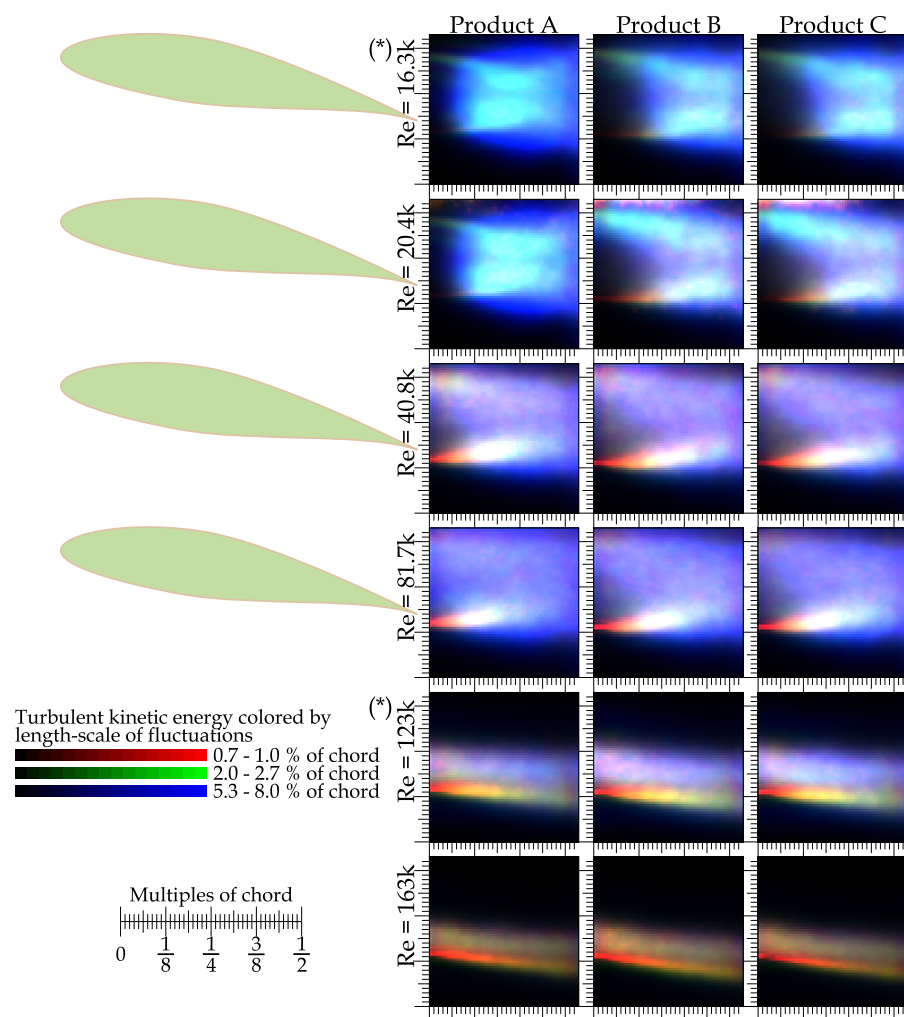


Figure A24. Turbulent kinetic energy colored by the length-scale of fluctuations producing it. TKE is divided into three channels: red channel represents smallest length-scale of fluctuations, green channel shows fluctuations of middle sizes, while the blue plays for the largest fluctuations. The colorscale among different color channels is normalized in such a way, that an ideal Kolmogorov turbulence would be displayed in shades of gray. Among different Re and variants, the colorscale is automatically adapted (for differences in amount of TKE look to Figure A23. Rows denoted by (*) are added.

465 Reynolds number of $1.63 \cdot 10^4$. The topology of the wake past product A at $Re = 2.04 \cdot 10^4$
 466 is more similar to the wake past all variants at $Re = 1.63 \cdot 10^4$. However, the plot of
 467 length-scale dependent TKE (Figure A24) reveals the difference between A and B, C
 468 even at smallest explored Reynolds number – wakes past all products are dominated
 469 by the largest length-scale at this Re; but the products B and C contain slightly larger
 470 amount of middle-scales and the maxima are better separated than past the A variant.

471 At Re between 4.1 and $8.2 \cdot 10^4$, the wake contains small-scale fluctuations in its
 472 pressure side (bottom part of figures), while in the suction side (upper in figures), it
 473 contains mostly fluctuations of larger scales being depleted by small scales, because
 474 the shear layer turbulence is *elder* here (the flow is detached close to the leading edge,
 475 then it develops dissipating energy at smallest scales due to the viscosity. Therefore,
 476 the smallest scales are depleted first). Thus the upper shear layer in Figure A24 is less
 477 localized and constructed by fluctuations of larger scales.

478 At $Re = 1.23 \cdot 10^5$, the flow is attached, as it has been discussed above and shown
 479 in Figure A21, but still the suction side of the wake contains significantly stronger signal

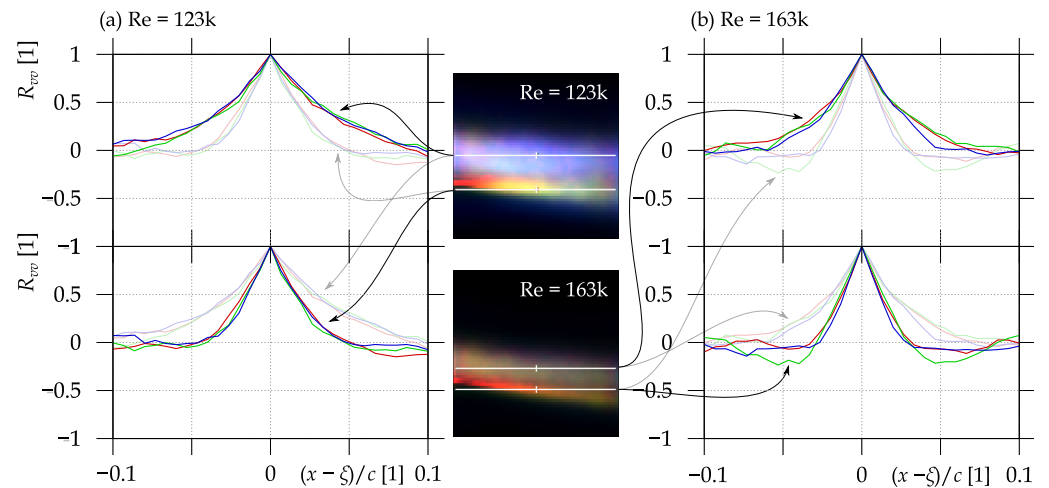


Figure A25. Autocorrelation function of fluctuating cross-stream velocity component v at two Reynolds numbers $1.23 \cdot 10^5$ (a,b) and $1.63 \cdot 10^5$ (c,d), where the flow is attached at $\alpha = 10^\circ$. First, the autocorrelation with reference point in the suction part of the wake (top) and second with reference point in the pressure part of the wake (bottom).

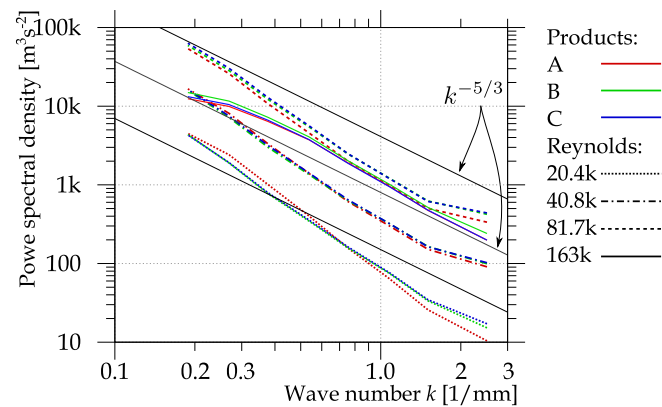


Figure A26. Spatial spectrum of the turbulent kinetic energy at $\alpha = 10^\circ$. Products are distinguished via colors, Reynolds number are by different linestyles. The thin black lines represent the $k^{-5/3}$ scaling.

480 from large-scale fluctuations, because the boundary layer at suction side (top) is less
 481 stable than at pressure side (bottom). At even higher Reynolds number, the boundary
 482 layer is more stable and the wake does not contain large-scale (blue) signal any more,
 483 however, there is an asymmetry between pressure and suction side. The autocorrelation
 484 function in Figure A25 shows the same – the coherent structures are significantly larger
 485 inside the suction part of the wake than in the pressure part. The different airfoil variants
 486 follow more or less the same pattern. only the variant B displays stronger periodic
 487 behavior at $Re = 1.63 \cdot 10^5$ in the autocorrelation with reference point in the pressure
 488 part of the wake, see Figure A25(d).

489 The spatial spectra, whose construction has been shortly described above or in
 490 more details in our previous publication [24], is plotted in Figure A26 for the angle of
 491 attack $\alpha = 10^\circ$. There is visible the contrast between the adhered flow at $Re = 1.63 \cdot 10^5$
 492 (displayed by solid lines) and the smaller velocities with stall. The largest Re contains
 493 less energy at largest scales, than the flow at *four times* smaller velocity! At middle scales,
 494 it contains comparable amount of energy as the flow at half the velocity. The power
 495 spectral density is steeper than the Kolmogorov scaling, except for the largest k , where

496 the instrumental noise appears. This scaling is even steeper at larger scales (which is not
497 valid for the largest Re).

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References

1. Frisch, U.; Kolmogorov, A.N. *Turbulence: the legacy of AN Kolmogorov*; Cambridge university press, 1995.
2. AIAA. *Guide for the Verification and Validation of Computational Fluid Dynamics Simulations*; G-077, AIAA, 1998. doi:10.2514/4.472855.001.
3. Drela, M. Airfoil Tools. <http://airfoiltools.com>. Accessed: 2021-04-20.
4. Moreno-Oliva, V.I.; Román-Hernández, E.; Torres-Moreno, E.; Dorrego-Portela, J.R.; Avendaño Alejo, M.; Campos-García, M.; Sánchez-Sánchez, S. Measurement of quality test of aerodynamic profiles in wind turbine blades using laser triangulation technique. *Energy Science and Engineering* **2019**, *7*, 2180–2192. doi:10.1002/ese3.423.
5. Drela, M. XFOIL: An Analysis and Design System for Low Reynolds Number Airfoils. *LOW REYNOLDS NUMBER AERODYNAMICS.PROC.CONF., NOTRE DAME, U.S.A., JUNE 5-7, 1989* **1989**, *54*, 1–12. doi:10.1007/978-3-642-84010-4.
6. Ravikovich, Y.; Kholobtsev, D.; Arkhipov, A.; Shakhov, A. Influence of geometric deviations of the fan blade airfoil on aerodynamic and mechanical integrity. *Journal of Physics: Conference Series*, 2021, Vol. 1891, p. 012042. doi:10.1088/1742-6596/1891/1/012042.
7. Klimko, M.; Okresa, D. Measurements in the VT 400 air turbine. *Acta Polytechnica* **2016**, *56*, 118. doi:10.14311/AP.2016.56.0118.
8. Uher, J.; Milcak, P.; Skach, R.; Fenderl, D.; Zitek, P.; Klimko, M. Experimental and Numerical Evaluation of Losses From Turbine Hub Clearance Flow. *Turbo Expo: Power for Land, Sea, and Air*, 2019, Vol. Volume 2B: Turbomachinery, p. 9. doi:10.1115/GT2019-90726.
9. Winthroth, J.; Seume, J.R. On the influence of airfoil deviations on the aerodynamic performance of wind turbine rotors. *Journal of Physics: Conference Series*, 2016, Vol. 753. doi:10.1088/1742-6596/753/2/022058.
10. Alsoufi, M.S.; Elsayed, A.E. Surface Roughness Quality and Dimensional Accuracy - A Comprehensive Analysis of 100Cost-Effective FDM 3D Printer. *Materials Sciences and Applications* **2018**, *09*, 11–40. doi:10.4236/msa.2018.91002.
11. Inkinen, S.; Hakkarainen, M.; Albertsson, A.; Södergård, A. From lactic acid to poly(lactic acid) (PLA): Characterization and analysis of PLA and Its precursors. *Biomacromolecules* **2011**, *12*, 523–532. doi:10.1021/bm101302t.
12. Zhang, C.; Zhao, H.; Gu, F.; Ma, Y. Phase unwrapping algorithm based on multi-frequency fringe projection and fringe background for fringe projection profilometry. *Measurement Science and Technology* **2015**, *26*. doi:10.1088/0957-0233/26/4/045203.
13. Li, F.; Stoddart, D.; Zwierzak, I. A Performance Test for a Fringe Projection Scanner in Various Ambient Light Conditions. *Procedia CIRP*, 2017, Vol. 62, pp. 400–404. doi:10.1016/j.procir.2016.06.080.
14. GOM, m. GOM Acceptance Test – Process Description, Acceptance Test according to the Guideline VDI/VDE 2634 Part 3, 2014.
15. Mendricky, R. Determination of measurement accuracy of optical 3D scanners. *MM Science Journal* **2016**, *2016*, 1565–1572. doi:10.17973/MMSJ.2016.12.2016183}.
16. Vagovský, J.; Buranský, I.; Görög, A. Evaluation of measuring capability of the optical 3D scanner. *Procedia Engineering*, 2015, Vol. 100, pp. 1198–1206. doi:10.1016/j.proeng.2015.01.484.
17. Du, W.; Zhao, Y.; He, Y.; Liu, Y. Design, analysis and test of a model turbine blade for a wave basin test of floating wind turbines. *Renewable Energy* **2016**, *97*, 414–421. doi:10.1016/j.renene.2016.06.008.
18. Gom, m. ATOS Core. <https://www.gom.com/en/products/3d-scanning/atos-core>. Accessed: 2021-04-20.
19. Krein, M.; Šmulian, V. On Regularly Convex Sets in the Space Conjugate to a Banach Space. *Annals of Mathematics* **1940**, *41*, 556–583. doi:10.2307/1968735.
20. Yanovych, V.; Duda, D.; horáček, V.; Uruba, V. Research of a wind tunnel parameters by means of cross-section analysis of air flow profiles. *AIP Conference Proceedings* **2019**, *2189*. doi:10.1063/1.5138636.
21. Yanovych, V.; Duda, D. Structural deformation of a running wind tunnel measured by optical scanning. *Strojnický Casopis* **2020**, *70*, 181–196. doi:10.2478/scjme-2020-0030.
22. Duda, D.; Yanovych, V.; Uruba, V. An experimental study of turbulent mixing in channel flow past a grid. *Processes* **2020**, *8*, 1–17. doi:10.3390/pr8111355.
23. Tropea, C.; Yarin, A.; Foss, J.F. *Springer Handbook of Experimental Fluid Mechanics*; Springer, Heidelberg, DE, 2007; p. 1557.
24. Duda, D.; Uruba, V. Spatial Spectrum From Particle Image Velocimetry Data. *ASME J of Nuclear Rad Sci.* **2019**, *5*. doi:10.1115/1.4043319.

25. Von Kármán, T. *Aerodynamics*; McGraw-Hill paperbacks : science, mathematics and engineering, McGraw-Hill, 1963.
26. Drela, M. Low-Reynolds Number Airfoil Design for the MIT Daedalus Prototype: A Case Study. *Journal of Aircraft* **1988**, *25*, 724–732.
27. Drela, M.; Giles, M. Viscous-Inviscid Analysis of Transonic and Low Reynolds Number Airfoils. *AIAA Journal* **1987**, *25*, 1347–1355.
28. Terra, W.; Sciacchitano, A.; Scarano, F. Aerodynamic drag of a transiting sphere by large-scale tomographic-PIV. *Exp. Fluids* **2017**, *58*. doi:10.1007/s00348-017-2331-0.
29. Terra, W.; Sciacchitano, A.; Scarano, F. Drag analysis from PIV data in speed sports. *Procedia Engineering* **2016**, *147*, 50–55. doi:10.1016/j.proeng.2016.06.188.
30. Ragni, D.; Oudheusden, B.W.; Scarano, F. Non-intrusive aerodynamic loads analysis of an aircraft propeller blade. *Experiments in Fluids* **2011**, *51*, 361–371. doi:10.1007/s00348-011-1057-7.
31. Oudheusden, B.W. PIV-based pressure measurement. *MEASUREMENT SCIENCE AND TECHNOLOGY* **2013**, *24*, 032001. doi:10.1088/0957-0233/24/3/032001.
32. Antonia, R.A.; Rajagopalan, S. Determination of drag of a circular cylinder. *AIAA Journal* **1990**, *28*, 1833–1834. doi:10.2514/3.10485.
33. Zhou, Y.; Alam, M.M.; Yang, H.; Guo, H.; Wood, D. Fluid forces on a very low Reynolds number airfoil and their prediction. *International Journal of Heat and Fluid Flow* **2011**, *32*, 329–339. doi:10.1016/j.ijheatfluidflow.2010.07.008.
34. Mohebi, M.; Wood, D.H.; Martinuzzi, R.J. The turbulence structure of the wake of a thin flat plate at post-stall angles of attack. *Experiments in Fluids* **2017**, *58*. doi:10.1007/s00348-017-2352-8.
35. Duda, D.; Yanovych, V.; Uruba, V. Wake Width: Discussion of Several Methods How to Estimate It by Using Measured Experimental Data. *Energies* **2021**, *14*, 4712. doi:10.3390/en14154712.
36. Eames, I.; Jonsson, C.; Johnson, P.B. The growth of a cylinder wake in turbulent flow. *Journal of Turbulence* **2011**, *12*, N39. doi:10.1080/14685248.2011.619985.
37. Romano, G.P. Large and small scales in a turbulent orifice round jet: Reynolds number effects and departures from isotropy. *International Journal of Heat and Fluid Flow* **2020**, *83*. doi:10.1016/j.ijheatfluidflow.2020.108571.
38. Lumley, J.L.; Newman, G. The return to isotropy of homogeneous turbulence. *J. Fluid Mech.* **1977**, *82*, 161–178. doi:10.1017/S0022112077000585.
39. Simonsen, A.J.; Krogstad, P.O. Turbulent stress invariant analysis: Clarification of existing terminology. *Phys. Fluids* **2005**, *17*, 088103. doi:10.1063/1.2009008.
40. Choi, K.; Lumley, J.L. The return to isotropy of homogeneous turbulence. *J. Fluid Mech.* **2001**, *436*, 59–84. doi:10.1017/S002211200100386X.
41. Duda, D.; Bém, J.; Yanovych, V.; Pavlíček, P.; Uruba, V. Secondary flow of second kind in a short channel observed by PIV. *European Journal of Mechanics, B/Fluids* **2020**, *79*, 444–453. doi:10.1016/j.euromechflu.2019.10.005.
42. Bém, J.; Duda, D.; Kovařík, J.; Yanovych, V.; Uruba, V. Visualization of secondary flow in a corner of a channel. AIP Conference Proceedings, 2019, Vol. 2189, pp. 020003–1–6. doi:10.1063/1.5138615.
43. Kundu, P.; Cohen, I.; Dowling, D. *Fluid Mechanics*, 6 ed.; Academic Press, 2016.
44. Duda, D.; Uruba, V. PIV of air flow over a step and discussion of fluctuation decompositions. AIP conference proceedings, 2018, Vol. 2000, p. 020005. doi:10.1063/1.5049912.
45. Agrawal, A. Measurement of spectrum with particle image velocimetry. *Experiments in Fluids* **2005**, *39*, 836–840. doi:10.1007/s00348-005-0018-4.
46. Agrawal, A.; Prasad, A. Properties of vortices in the self-similar turbulent jet. *Experiments in Fluids* **2002**, *33*, 565–577. doi:10.1007/s00348-002-0507-7.
47. Duda, D.; La Mantia, M.; Skrbek, L. Streaming flow due to a quartz tuning fork oscillating in normal and superfluid He 4. *Physical Review B* **2017**, *96*. doi:10.1103/PhysRevB.96.024519.
48. Kolmogorov, A.N. Dissipation of Energy in the Locally Isotropic Turbulence. *Proceedings of the Royal Society A: Mathematical, Physical and Engineering Sciences* **1991**, *434*, 15–17. doi:10.1098/rspa.1991.0076.
49. Jiang, M.T.; Law, A.W.K.; Lai, A.C.H. Turbulence characteristics of 45 inclined dense jets. *Environmental Fluid Mechanics* **2018**, pp. 1–28. doi:10.1007/s10652-018-9614-8.
50. Barenghi, C.F.; Sergeev, Y.A.; Baggaley, A.W. Regimes of turbulence without an energy cascade. *Scientific Reports* **2016**, *6*, 1–11, [1609.09705]. doi:10.1038/srep35701.
51. Harun, Z.; Abbas, A.A.; Lotfy, E.R.; Khashehchi, M. Turbulent structure effects due to ordered surface roughness. *Alexandria Engineering Journal* **2020**, *59*, 4301–4314. doi:10.1016/j.aej.2020.07.035.
52. Bourgoïn, M.; Baudet, C.; Khariche, S.; Mordant, N.; Vandenberghe, T.; Sumbekova, S.; Stelzenmuller, N.; Aliseda, A.; Gibert, M.; Roche, P.; Volk, R.; Barois, T.; Caballero, M.L.; Chevillard, L.; Pinton, J.; Fiabane, L.; Delville, J.; Fourment, C.; Bouha, A.; Danaila, L.; Bodenschatz, E.; Bewley, G.; Sinhuber, M.; Segalini, A.; Örlü, R.; Torrano, I.; Mantik, J.; Guariglia, D.; Uruba, V.; Skala, V.; Puczyłowski, J.; Peinke, J. Investigation of the small-scale statistics of turbulence in the Modane S1MA wind tunnel. *CEAS Aeronautical Journal* **2018**, *9*, 269–281. doi:10.1007/s13272-017-0254-3.
53. Kraichnan, R.H. Inertial ranges in two-dimensional turbulence. *Physics of Fluids* **1967**, *10*, 1417–1423. doi:10.1063/1.1762301.
54. Schulz-DuBois, E.O.; Rehberg, I. Structure Function in Lieu of Correlation Function. *Applied Physics* **1981**, *24*, 323–329.
55. Kubíková, T. The air flow around a milling cutter investigated experimentally by particle image velocimetry. AIP Conference Proceedings, 2021, Vol. 2323, pp. 030006–1–8. doi:10.1063/5.0041860.

56. Azevedo, R.; Roja-Solórzano, L.R.; Bento Leal, J. Turbulent structures, integral length scale and turbulent kinetic energy (TKE) dissipation rate in compound channel flow. *Flow Measurement and Instrumentation* **2017**, *57*, 10 – 19. doi:<https://doi.org/10.1016/j.flowmeasinst.2017.08.009>.